# LIGHT-DUTY DIESEL EMISSION CORRECTION FACTORS FOR AMBIENT CONDITIONS



Environmental Sciences Research Laboratory
Office of Research and Development
U.S. Environmental Protection Agency
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# LIGHT-DUTY DIESEL EMISSION CORRECTION FACTORS FOR AMBIENT CONDITIONS

by

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> Task 5 Interim Report Contract No. 68-02-1777

> > 'Project Officer

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#### FORWARD

This document presents completed work on one phase (out of five) of a large contract effort characterizing diesel engine emissions. This particular segment is, therefore, an interim report of findings by Southwest Research Institute relative to the value of humidity correction factors needed for testing diesel-powered passenger cars for NO\_ emissions.

Ambient air temperature, humidity, and barometric pressure influence the emission rates of pollutants from passenger cars. For example, in cold weather gasoline engines are slow to warm up, carburetor chokes remain closed longer, and hydrocarbon emissions are elevated. The emission rate of NO from a passenger car is especially sensitive to humidity. The reason is well known; namely, the higher the water vapor concentration in the engine charge, the lower the effective fuel-air mixture density must be. Thus, high humidity produces low rates of heat release, low cylinder gas temperatures, and hence low NO.

In emissions certification, ambient conditions can be held constant and, thus, all cars can be tested fairly relative to one another. However, not all manufacturers have the extremely expensive equipment necessary to maintain humidity and temperature constant in a dynamometer cell. Furthermore, vehicles are operated under a wide range of ambient conditions. It is, therefore, important to predict emissions under conditions other than the standard ones. Hence, statistical correction factors are needed.

At the outset of this work, correction factors were available for gasoline engine cars operated over the current Federal urban cycle and for heavy duty diesel engines at constant speeds. However, no such factors were available for diesel-powered passenger cars, a class of vehicles projected to be of increasing importance as future fuel economy goals are pursued. Consequently, the task of developing these needed factors was contracted to Southwest Research Institute.

The authors report new values for NO -humidity correction factors considerably smaller than those used for gasoline-powered cars. This fact represents a small credit for NO emissions from diesel versus gasoline engines operating in hot, humid climates. Since relatively high-powered cars were not available for this study, there is probably a danger inherent in projecting these factors to high-power-to-weight

#### ABSTRACT

Since emission measurements from passenger cars are performed at one standard set of ambient conditions and since emission rates of HC, CO, and NO are sensitive to temperature and humidity, it is necessary to determine the influence of ambient conditions on emissions from major classes of vehicles. Although such information has been available for gasoline engine powered cars for sometime, no such data were available for diesel powered passenger cars.

This report indicates that diesel HC and CO emissions are relatively insensitive to ambient conditions. Diesel NO emissions, however, are sensitive to humidity but to a smaller extent than gasoline engines. Humidity correction factors for NO emissions also appear to vary with vehicle power-to-weight ratios and are greater for higher powered vehicles.

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ratio vehicles such as the new V-8 engines due in the fall of 1977. In fact, the authors point out that the one six-cylinder vehicle tested was much more humidity-sensitive than the four-cylinder models, and plausible reasons are given for this effect.

No significant temperature or humidity effects for hydrocarbon or CO emissions were found. This is probably due to the quick warmup of diesels relative to gasoline engines. Since the range of barometric pressures available in San Antonio was small, the current results are not necessarily applicable to high altitude, low station pressure areas such as the Rocky Mountain States.

Dr. Ronald L. Bradow Project Officer

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# SECTION 1

#### INTRODUCTION

Light-duty, diesel-powered vehicles were included under Federal exhaust emission standards beginning with the 1975 model year, (1) recognizing that their U.S. sales volume was likely to become appreciable in the near future. At that time, as is still the case today, all diesel-powered, light-duty vehicles available to the consumer were either of foreign manufacture or were equipped with engines of foreign manufacture. These vehicles have traditionally been powered by relatively small engines, having displacements of 3 liters (183 in<sup>3</sup>) or less. Emission test procedures for light-duty diesels have been as similar as possible to those for gasoline-powered vehicles, and emission standards for the two engine types used in light-duty vehicles have been (and probably will remain) the same.

At this writing, it appears that the diesel-powered automobile is on the threshold of a relative "population explosion" in the United States. Concern over fuel economy is one of the driving forces behind this predicted expansion, but another is certainly a desire by auto manufacturers to secure a competitive advantage by offering the consumer something novel. The vehicles which will create the boom, if it comes within the next two years or so, will be the Volkswagen and Oldsmobile diesels. Anticipating this situation, it becomes more important to refine existing emission test procedures, providing additional assurance that present and future emission standards will in fact achieve air quality goals.

To date, with the exception of continuous HC sampling and integration, calculation procedures for light-duty diesel FTP's have been the same as those for light-duty gasoline FTP's. EPA recognized in the regulations for light-duty diesels,  $^{(1)}$  however, that the NO $_{\!X}$  correction factor for intake air humidity ( $K_h$ ) could be modified as necessary pending the availability of test data. This report contains the information required to make decisions on factors for correction of light-duty, diesel-powered vehicle emissions to standardized ambient conditions. These decisions should help to place measured diesel emissions values on a firmer base, thereby providing greater accuracy in comparison of environmental hazards associated with gasoline- and diesel-powered vehicles.

#### SECTION 2

#### CONCLUSIONS

l. Combined data from the four light-duty diesel test vehicles, using a linear model, yielded the following humidity (H) correction factor for  $NO_X$  (same for two equal slope/equal intercept variations):

$$K_{h} = \frac{1}{1 - 0.00217 \text{ (H-75)}}$$
,

where H = humidity in grains  $\rm H_2O/lb_m$  dry air. The equations on which the factor is based displayed correlation coefficients ( $\rm r^2$ ) of 0.569 ("normalized" data) and 0.566 ("standardized" data) with the combined emissions data. This factor is very similar to that originally used for correction of  $\rm NO_X$  emissions from heavy-duty diesels. (3)

2. On the average, a quadratic equation in H (one containing H and  ${\rm H}^2$  terms) correlated better with  ${\rm NO_X}$  data than did either a linear equation in H alone or one linear in H and temperature (T). For data on individual vehicles, correlation coefficients ( ${\rm r}^2$ ) for the quadratic averaged 0.653, those for the linear in H averaged 0.570, and those for the linear in H and T averaged 0.594. The factor computed using quadratics in H (average of coefficients for equations using both "standardized" and "normalized" data) is

$$K_{h} = \frac{1}{1 - 0.00228 \text{ (H-75)} + (1.86 \times 10^{-5}) \text{ (H-75)}^2}$$

and the  $r^2$  of the quadratic factor is 0.616 for "normalized" data and 0.613 for "standardized" data.

- 3. In addition to the combinations of "independent" variables already mentioned (H alone, H with T, and H with  $\rm H^2$ ), emissions were also regressed against: T alone; T and T<sup>2</sup>; H, T, and HT; and H, T, H<sup>2</sup>, and T<sup>2</sup>. For  $\rm NO_X$ , correlations were either worse than for the linear and/or quadratic in H, or else the additional complication of introducing more variables could not be justified in terms of improved correlation. For the other emissions (HC and CO), results were too mixed and/or correlations were too poor to justify computation of correction factors from the equations.
- 4. Emissions of HC and CO from the International 100 pickup truck equipped with Perkins 6.247 engine were more strongly dependent on humidity and temperature than those from the other vehicles. No facts are available to explain this result, but it may be related to the much lower specific loading (kg vehicle mass per available engine kW) of the Perkins engine as

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compared to the others (lower specific loading would mean lower overall F/A ratios for this vehicle).

- 5. Combination of  $\mathrm{NO}_{\mathbf{X}}$  emission values for the four vehicles by "normalizing" them appeared to yield good data for computation of a final correction factor. This process eliminated the effect of differing  $\mathrm{NO}_{\mathbf{X}}$  emission magnitudes among the test vehicles by transforming the data to ratios of "as measured" values versus "best predicted" values at a standard humidity among the four test vehicles.
- 6. Use of a greater number of test vehicles would be desirable for any future research aimed at improving the statistical basis for light-duty diesel emission correction factors.

#### SECTION 3

# VEHICLES, FUEL, AND TEST INSTRUMENTATION

Each topic of this section is treated in a separate subsection for clarity. Vehicle parameters and specifications are outlined first, followed by test fuel specifications and requirements. Instrumentation used for testing and analysis is discussed to conclude the section.

#### TEST VEHICLES

The four light-duty, diesel-powered vehicles used for test purposes were a Datsun 220C, an International pickup with Perkins 6.247 engine, a Mercedes 240D, and a Peugeot 504D. These vehicles are shown in Figures 1 through 4 for documentation, and descriptions of them are given in Table 1. It was planned initially to use five test vehicles, but the fifth one was not available when needed. The decision to proceed with only four vehicles, but to conduct more tests per vehicle than had been planned, was approved by the Project Officer.

The particular test vehicles used reflected availability of vehicles for EPA programs at the time testing began more strongly than they reflected the population of diesel-powered, light-duty vehicles. The Mercedes 240 and Peugeot 504 were the only diesel automobiles on the U.S. consumer market when testing began, so to that extent they could be considered representative. The other two vehicles, however, were research prototypes as far as the U.S. market was concerned. Loaded vehicle weights ranged from about 1400 to 2000 kg, and engine size ranged from 2.1 to 4.1 liters. All the engines were of the indirect injection, naturally-aspirated type, with similar injection systems (Bosch and Bosch-liscensed) and compression ratios between 21.0 and 22.2. Each vehicle was equipped with a 4-speed manual-shift transmission.

#### TEST FUEL PROPERTIES

All four vehicles were operated on Type 2-D emissions test fuel as specified in Federal regulations. (1) Inspection results on the particular fuel batch used, EM-238-F, are given in Table 2 along with required specifications and "national average" properties for comparison. The test fuel was well within Federal specifications for all properties except end point, at which it was coincident with the upper limit. As compared to a "national average" No. 2 fuel, the test fuel contained more sulfur and somewhat more high-boiling material. Although no hydrocarbon composition data were available in the survey data, (2) it is likely that the test fuel contained more aromatics than an average No. 2 fuel.



Figure 1. Datsun 220C.



Figure 2. International 100 with Perkins 6.247 engine.



Figure 3. Mercedes 240D.



Figure 4. Peugeot 504D.

σ

TABLE 1. DESCRIPTION OF TEST VEHICLES

Vehicle Model	Datsun 220C	International 100	Mercedes 240D	Peugeot 504D
Engine Model (if different)	Nissan SD22	Perkins 6.247	OM616	XD90
V.I.N. Engine No. (if different)	QL230-103467	4H1CODHB23906	11511710066208	504A90-2034350
	SD22-116440	247J1042	616916-10-052895	X203043508
Body Type. Loaded Weight, kg (lbm) <sup>a</sup> Inertia Equivalent, kg (lbm)	4 door sedan	pickup truck	4 door sedan	4 door sedan
	1551 (3419)	1982 (4370)	1492 (3289)	1402 (3091)
	1588 (3500)	2041 (4500)	1588 (3500)	1361 (3000)
Transmission	4 speed manual	4 speed manual	4 speed manual	4 speed manual
Displacement, 1 (in <sup>3</sup> ) Cylinders Power, kW (hp) @ rpm Injection System Combustion Chamber Compression Ratio	2.16 (132.1)	4.06 (247.7)	2.40 (146.7)	2.11 (128.9)
	4	6	4	4
	52.2 (70) @ 4000	91.0 (122) @ 4000	46.2 (62) @ 4350	48.5 (65) @ 4500
	Kiki	Kiki	Bosch	Bosch
	prechamber	prechamber	prechamber	prechamber
	22.0	21.1	21.0	22.2
Distance on Vehicle, kmb	19,861	17,830	4,677	4,694

 $<sup>^{\</sup>rm a}$  curb weight plus 136 kg (300  $1b_{\rm m})$  b at end of tests

TABLE 2. PROPERTIES OF TEST FUEL, FEDERAL SPECIFICATIONS, AND "NATIONAL AVERAGES" FOR COMPARISON

Fuel Type	2D Test Fuel	Federal 2D	"National Average"
Fuel Code	EM-238-F	Specification	No. 2 <sup>a</sup>
Density, g/ml	0.845	b	b
Gravity, °API	36.0	33 - 37	35.7
Cetane (D976) Viscosity, CS (D445) Flash Point, °C (°F) Sulfur, wt. % (D1266)	48.6	42 - 50	49.3
	2.65	2.0 - 3.2	2.71
	94.(202)	54 (130) minimum	b
	0.35	0.2 - 0.5	0.249
FIA:    aromatics, %    olefins, %    saturates, %	29.8 1.6 68.6	27 (minimum) b b	b b
Distillation (D86): IBP, °C (°F) 10% pt., °C (°F) 50% pt., °C (°F) 90% pt., °C (°F) EP, °C (°F)	192 (378)	171-204 (340-400)	190 (374)
	213 (415)	204-238 (400-460)	221 (430)
	257 (495)	243-282 (470-540)	261 (502)
	312 (593)	288-321 (550-610)	307 (585)
	349 (660)	304-349 (580-660)	333 (632)
Carbon, wt. % Hydrogen, wt. % Nitrogen, wt. %	86.8 12.9 0.005	b	b b

 $<sup>^{\</sup>rm a}$  average of five regional averages, 1976 ERDA Diesel Fuel Survey  $^{\rm (2)}$  ,  $\frac{\rm not}{\rm no}$  sales-weighted  $^{\rm no}$  specification or no data

A Type 1-D diesel fuel is specified alongside the Type 2-D fuel in the heavy-duty and light-duty emission regulations, (3,1) in case a given manufacturer requires No. 1 to be used in its engines. For the test vehicles and for other market entries anticipated, however, No. 2 diesel fuel will probably continue to be recommended. The main reason for the more widespread use of No. 2 fuel is economy. Its price per unit volume is equal to or lower than No. 1 fuel, while having considerably greater density (and proportionately higher energy content) per unit volume. The only foreseeable circumstance which would move fuel usage for diesel cars toward No. 1 fuel would be a dramatic increase in urban diesel smoke and/or odor complaints as the light-duty diesel population increases.

# INSTRUMENTATION AND TEST EQUIPMENT

The four diesel-powered vehicles used for test purposes were operated on a standard 2-roll chassis dynamometer, in this instance a Clayton Model CT-200 which had been modified to EC-50 configuration. This dynamometer used a 37.3 kW (50 hp) water brake absorber and a belt-driven variable inertia system to simulate road operation. Inertia and power settings were based on vehicle weight and were set according to Federal procedure. (1) For test purposes, the rear tires of the vehicles were inflated to 3.16 kg/cm<sup>2</sup> (45 psig) to minimize deflection on the rolls. The Datsun 220C vehicle shown in Figure 5 was operating on the chassis dynamometer.

Figure 5 also shows the position of the auxillary cooling fan in front of the vehicle, producing an air flow of approximately 2.36 m³/sec (5000 ft³/min). Sampling or measurement points for all the air analysis instrumentation were located within 0.6 m (2 ft) of the inlet plane of this fan. The air instrumentation included two air (dry bulb) temperature thermocouples, one forced air psychrometer, one electronic hygrometer, and a dewpoint-measuring device. Figure 6 shows another view of the instruments and sampling/measurement points with the psychrometer at upper left, perforated relative humidity/dry bulb temperature sensor for the electronic hygrometer at center, bare-tip thermocouple at bottom center, and dewpoint instrument at bottom right. The white object near top center is a small funnel to which the dewpoint instrument's sample line was attached. Another view of the area behind the fan is given by Figure 7.

Of the humidity- and temperature-measuring instruments noted above, only the electronic hygrometer output and the two dry-bulb temperatures were recorded on a continuous basis. The other instruments were monitored manually, and readings were taken from them at intervals of 2 to 5 minutes during each test. Yet another source of data was the National Weather Service, from which humidity data were obtained on an hourly basis during the days and times when tests were being conducted. The Weather Service data were not intended as primary information to be used in a statistical sense, but rather as corroboration of data obtained by our direct measurements. Accuracy of all the measurements and correlations between systems will be discussed later in the report.

Measurement of CO,  $NO_X$ , and  $CO_2$  gaseous emissions was accomplished using a constant-volume sampler (CVS) and a set of low-concentration gas analyzers



Figure 5. Datsun 220C test vehicle on dynamometer, with humidity and temperature measuring equipment.



Figure 6. Details of air sampling and measurement points for humidity and temperature.

to read the diluted (bag) emission concentrations. Hydrocarbon emissions were sampled just after dilution occurred (prior to entry into the CVS), and were analyzed by a heated FID on a continuous basis. An electronic integrator provided the means of extracting an average value from the FID output. The CVS used is shown in Figure 8; and the heated FID detector/oven, control unit, chart recorder, and integrator are shown in Figure 9. Instruments used for measurement of bag concentrations (called the "bag cart") are shown in Figure 10. This cart contains a chemiluminescent NO<sub>X</sub> analyzer, an NDIR CO<sub>2</sub> analyzer, and two low-range NDIR CO analyzers (one long-path and one standard).

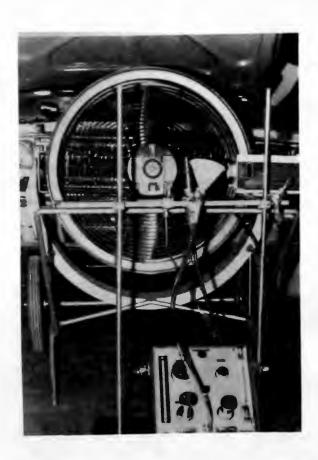


Figure 7. Second view of air sampling and measurement points.





Figure 8. Constant-Volume Sampler (CVS) used for light-duty diesel exhaust sampling.

Figure 9. Details of heated hydrocarbon analysis equipment.



Figure 10. Instrumentation for measurement of dilute (bag) emission concentrations.

#### SECTION 4

#### EXPERIMENTAL PLAN AND TEST PROGRAM DETAILS

This section deals first with the experimental plan designed to gather meaningful data about effects of ambient conditions on light-duty diesel emissions. The second subsection covers the details of the test program as it actually occurred and the effects of deviations from the original plans. All this information constitutes the foundation on which the results and conclusions of the program are based.

#### EXPERIMENTAL PLAN

Following submittal of the Contract Work Plan early in the program, efforts began to assemble the apparatus required for control of intake air temperature and humidity. A problem was discovered with the planned approach, however, because it was to control properties of the engine intake air only rather than the ambient. The fallacy in the original line of thought was that "ambient" data were to be taken in a controlled airstream leading to the vehicle air intake, rather than in a totally-controlled ambient (the room). Once this problem had been thoroughly discussed, it was decided to use naturally-occurring humidity conditions with control on room temperature only. A revised Work Plan was submitted to document this change. Original plans also called for a five-vehicle test program, but it was later agreed to utilize four vehicles (with a greater number of tests per vehicle) because the fifth vehicle could not be supplied for the program. It was recognized that a greater number of test vehicles would produce more representative statistics on light-duty diesels, but other vehicles were simply not available for test purposes.

The Contract "TEST SCHEDULE", with computational corrections as necessary, is reproduced in Table 3. The specified tolerance on relative humidity for each test was ± 2 percent. It was decided that for test purposes, a slightly different set of temperatures would be employed, namely 68, 77, and 86°F (20, 25, and 30°C). The reason for this change was that light-duty FTP regulations call for test temperatures from 68 to 86°F (20 to 30°C). With this minor modification, the relative humidity portion of Table 3 was recomputed and now appears as Table 4. These conditions in Table 4 were those sought (or an approximation thereof) during the test program. Variables actually used to decide on the worth of running at a given set of ambient conditions were (or were calculated from) original independent variables; and they were temperature and specific humidity expressed in grains H<sub>2</sub>O/lb<sub>m</sub> dry air.

TABLE 3. CONTRACT "TEST SCHEDULE"

Note: Replications in parentheses

	Humidity exp	]		<del></del>		
grains H <sub>2</sub> O/				Relative	e humidity	(%) at
wt. % H <sub>2</sub> O	vol. % H <sub>2</sub> O	lb <sub>m</sub> dry air	Pd/Pw	65°F	75°F	85°F
0.50	0.802	35.2	0.992	37.8(2)	26.9(2)	
0.75	1.201	52.9	0.988	56.5(2)	40.2(3)	
1.00	1.598	70.7	0.984	75.3(2)	53.5(3)	38.6(2)
1.25	1.995	88.6	0.980		66.8(3)	48.2(2)
1.50	2.390	106.6	0.976		80.0(2)	57.7(2)
1.75	2.785	124.7	0.973			67.2(2)
2.00	3.178	142.9	0.969			76.7(2)

TABLE 4. REVISED TEMPERATURE AND RELATIVE HUMIDITY VALUES

Humidity as	Relative humidity (%) at				
grains H <sub>2</sub> O/lb <sub>m</sub> dry air	68°F (20°C)	77°F (25°C)	86°F (30°C)		
35.2 ( 5.03) <sup>a</sup>	34.0	25.1			
52.9 ( 7.56)	51.0	37.6			
70.7 (10.1 )	67.8	50.1	37.4		
88.6 (12.7 )		62.5	46.6		
106.6 (15.2 )		74.9	55.9		
124.7 (17.8 )			65.1		
142.9 (20.4 )			74.3		

a values in parentheses in g H2O/kg dry air

Noting the number of replications specified for each set of conditions given in Table 3, the original test plan called for 29 FTP's per vehicle (145 FTP's total). It was also requested that the order of the tests be randomized on a daily basis, so that the vehicles would not be tested in the same order all the time. This request was complied with by preparing a randomized daily sequence based on a table of random digits. It does not seem necessary to reproduce this sequence as a separate item, since it can be deduced readily from general data tabulations (including dates and run numbers) which are presented later in the report.

#### DETAILS OF THE TEST PROGRAM

The decision to use naturally-occurring humidity conditions so that room ambients could be the variables of record resulted in a somewhat different set of ambient conditions for tests on each vehicle. Room temperatures were sometimes increased by adding heat, but decreases in temperatures were not attempted due to the probability of moisture removal in the air conditioners. In a few cases, tests were conducted using a steam generator in the room to maintain humidity above ambient. Moisture addition was necessary only for some of the higher specific humidity conditions. Both humidity and temperature remained essentially constant during individual tests.

Rather than refer again to general data tabulations for actual test ambient conditions, these data are plotted in Figures 11a through 11d as compared to the planned set of ambient conditions. These graphs should make visualization of the comparison easier than would a simple tabulation. Although the humidity points did not fall exactly on the planned values in most cases, the range and distribution of points achieved should be satisfactory from a statistical standpoint. Since humidity points had to be accepted essentially as they occurred naturally, a larger number of tests had to be performed than was planned initially. A total of 174 valid tests were conducted on the four vehicles, which compares to a total of 145 planned tests on five vehicles. The tests were split quite evenly among the vehicles; with 44 being conducted on the Datsun 220C, 45 on the International/Perkins 6.247, 43 on the Mercedes 240D, and 42 on the Peugeot 504D.

Measures used to help control data quality throughout the test program included CVS propane checks, dynamometer calibrations, and NO, converter checks. Data from each day of testing were tabulated and graphed to determine whether or not any investigation should be conducted for processing errors. The analysis instrumentation was fully calibrated on a monthly basis with gases named by EPA's Ann Arbor laboratory or with gases traceable to them. This instrumentation and these gases were further used to cross-check four  $NO_{\mathbf{y}}$  calibration gases sent to SwRI by the Project Officer. The results of this cross-check are given below in Table 5; and it is apparent that agreement is quite good in the lower concentrations, but somewhat less satisfactory as the concentrations increase (disagreements up to about 4 percent). Almost all the NOx concentrations analyzed during this program were in the range of 30 to 50 ppm (dilute sample bags). No reasons have been identified as yet for the apparent calibration differences indicated in Table 5. While absolute accuracy is important for all the emissions data, it probably is just as important for the ambient data.

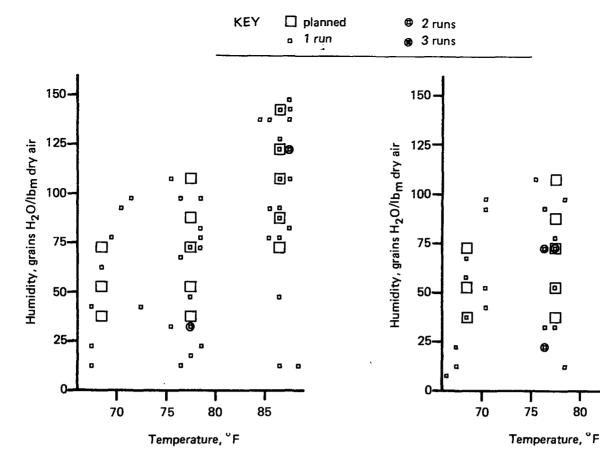


FIGURE 11a. PLANNED AND ACTUAL AMBIENT CONDITIONS, DATSUN 220C

FIGURE 11b. PLANNED AND ACTUAL AMBIENT CONDITIONS, PERKINS 6.247

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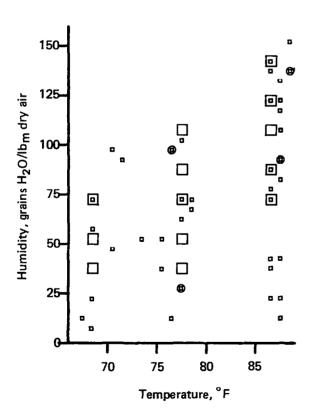


FIGURE 11c. PLANNED AND ACTUAL AMBIENT CONDITIONS, MERCEDES 240D

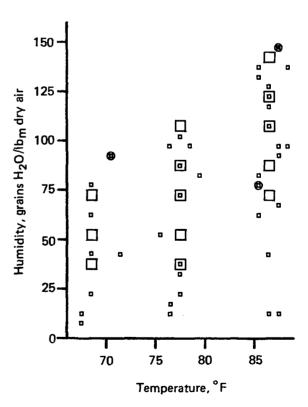


FIGURE 11d. PLANNED AND ACTUAL AMBIENT CONDITIONS, PEUGEOT 504D

TABLE 5. RESULTS OF NO CALIBRATION GAS CROSS-CHECK

	Concentration in ppm by analyzing laboratory							
Cylinder	EPA-Research SwRI - SwRI - SwRI - EPA-Research							
number	Triangle Park	first check	second check	Arbor				
MM-2784	24.0	23.6		24.07				
MM-2892	89.0	87.5		88.87				
MM-2930	279.5	271.	273.	265.7				
MM-2890	534.	516.	511.	505,2_				

a checked against NBS calibration gases

Data on ambient conditions were recorded continuously near the inlet of the vehicle cooling fan, which is the recommended location for such measurements. These data (dry-bulb temperature and relative humidity) were also integrated electronically, and they were read manually at intervals of approximately 2 minutes. Ambient temperature and dewpoint temperature were measured manually at somewhat longer intervals (about 5 minutes), and both wet- and drybulb temperatures were recorded on a similar schedule. Relative humidity data from the electronic hygrometer proved to be most reliable of the three humiditymeasuring measurements. This instrument was also calibrated periodically according to ASTM recommended practice E104-51. The dewpoint instrument was used for all tests during which it operated properly, but it had to be repaired several times during the program. It was also 'discovered that data taken using the psychrometer were inaccurate unless an inordinate amount of time was devoted to its care and maintenance; so it was eliminated early in the program, and no psychrometer data appear in this report. The care necessary to obtain accurate psychrometer data has been discussed in the literature in some detail. (4)

It has already been noted that variation in humidity during each particular run was requested to be  $\pm$  2 percent relative humidity or less, as referred to the mean. This tolerance is, therefore, a function of several variables. It ranges from  $\pm$  2.1 grains  $\rm H_2O/lb_m$  dry air ( $\pm$  0.29 g  $\rm H_2O/kg$  dry air) at 68°F (20°C) and 28.80 in Hg to  $\pm$  4.0 grains  $\rm H_2O/lb_m$  dry air ( $\pm$  0.57 g  $\rm H_2O/kg$  dry air) at 86°F (30°C) and 29.78 in Hg. These acceptance bands are quite reasonable in most cases, especially for the situation in which humidity is being controlled. To determine the acceptance band for a given test, the mean relative humidity was first computed from mean specific humidity, temperature, and atmospheric pressure. Relative humidity was then permitted to vary  $\pm$  2 percent, and specific humidity was calculated for the extremes. Some runs were included in the data base which did not quite meet the  $\pm$  2 percent R.H. criterion. The additional criteria which these latter runs did meet included:

- relative humidity range within approximately + 5 percent of mean,
- absence of variations which would invalidate time-averaged mean,
- absence of anomalous emissions data.

#### SECTION 5

#### RESULTS, ANALYSIS, AND CORRECTION FACTOR COMPUTATIONS

The first part of this section is devoted to presentation of the results in summary form and to statistical analysis of the emission and ambient data. The second and last subsection covers computation of correction factors for  $NO_{\mathbf{x}}$  at non-standard humidity conditions.

#### RESULTS AND STATISTICAL ANALYSIS

The results of this Task were those which might have been expected after a review of previous work on studies of the relationship between emissions and ambient conditions.  $^{(5,6)}$  Some of the trends that were observed are summarized below:

- increases in humidity were associated with substantial decreases in  $\mathrm{NO}_{\mathbf{x}}$  for all four vehicles
- changes in temperature were associated with relatively minor changes in  $NO_{\mathbf{x}}$  for all four vehicles
- changes in humidity and temperature were associated with relatively minor changes in production of hydrocarbons and CO from three of the four test vehicles.

The main objective of this task, in addition to confirming existence of the above trends, was to compute the correction factors needed to correct measured emission values to those that would be expected at standard ambient conditions. All the emission correction factors which have been adopted for use in Federal emission regulations in the past are different from one another (1,3,7), so it could not be assumed, without testing, that the light-duty diesel could legitimately use one of these other factors.

Emission and ambient data, contained in Appendix A, were gathered on 174 valid FTP runs. In order to determine if any linear relationships existed between the supposed "independent" variables (humidity by three methods, temperature, and atmospheric pressure), correlation coefficients were calculated for each pair of variables. It was expected that high correlations would exist among the humidity values determined by the three methods, that temperature would be only weakly dependent on atmospheric pressure and humidity, and that humidity (regardless of method) would be essentially independent of atmospheric pressure. The correlation coefficients (r) actually calculated are presented in Table 6 and indicate the strengths of the relationships between the variables.

TABLE 6. CORRELATION COEFFICIENTS (r) BETWEEN VARIABLES OF RECORD

Independent variables	Hygrometer humidity, gr/lb <sub>m</sub>	Dewpoint humidity, gr/lbm	Weather service humidity, gr/lb <sub>m</sub>	Temp. °F
dewpoint humidity, grains $H_2O/lb_m$ dry air	0.965			
weather service humidity, grains ${\rm H_2O/lb_m}$ dry air	0.925	0.893		
temperature, °F	0.492	0.552	0.251	
atmospheric pressure, in Hg	-0.746	-0.778	-0.710	-0.348

Although all correlations were statistically significant (p < 0.05 using "t" statistics), those between "independent" variables were in accord with expectations except the relationships between humidity and atmospheric pressure. All three humidity values correlated more strongly with atmospheric pressure than expected. This result was created by a local situation, namely that decreases in atmospheric pressure are frequently followed by southerly winds carrying humid air from the Gulf of Mexico. Different correlations probably exist in other areas due to differences in topography, latitude, and proximity to large bodies of water.

Since all the humidity values were highly correlated ( $r \ge 0.89$ ) and since humidity as measured with the electronic hygrometer was the most reliable result from the methods employed, the hygrometer values alone were used in the statistical analysis. Temperature was the second variable chosen due to its weak relationship with humidity and atmospheric pressure. Finally, atmospheric pressure was not considered as an important ambient variable due to its high correlation ( $r \le -0.71$ ) with humidity and the expected overlap of information that would result if both these variables were included. Thus, only temperature and hygrometer humidity were chosen to be used as the independent variables in the analyses described below.

Generalized linear equations were determined utilizing emissions (HC, CO, or  $NO_X$ ) as the dependent variable and humidity (H) and temperature (T) as the independent variables. Linear as well as polynomial regressions were calculated for each vehicle and each pollutant. The equations were of the following forms:

(1) 
$$\hat{E} = b_0 + b_1 X$$

and (2) 
$$\hat{E} = b_0 + b_1 X + b_{11} X^2$$

where  $\hat{E}$  = emission value predicted by the regression equation

bo = constant term

b<sub>1</sub> = regression coefficient for linear effect of variable X

 $b_{11}$  = regression coefficient for quadratic effect of variable X

X = independent variable (humidity or temperature).

The coefficients obtained using equations (1) and (2) with humidity and temperature as the independent variables are contained in Appendix B. Appendix C contains computer printouts of stepwise multiple regressions conducted on individual vehicles, including analysis of variance and summary tables.

Table 7 consists of correlation comparisons between the linear and quadratic fits of humidity to emissions. Using the coefficients of determination  $(r^2)$  as a criterion, the best fits are between  $NO_X$  and humidity (average  $r^2$  = 0.653 for quadratic fit), although all emissions are strongly associated with humidity for the International-Perkins.

		Coefficie	Coefficients of determination (r <sup>2</sup> )by vehicle					
Dependent Variable	Independent Variable(s)	Datsun	Int'l. Perkins	Mercedes	Peugeot	Average		
HC ·×	H	0.020	0.576	0.000	0.078	0.168		
	H, H <sup>2</sup>	0.085	0.584	0.012	0.127	0,202		
	Improvement	0.065	0.008	0.012	0.049	0.034		
со	H	0.050	0.697	0.002	0.000	0.187		
	H, H <sup>2</sup>	0.051	0.724	0.054	0.002	0.208		
	Improvement	0.001	0.027 <sup>a</sup>	0.052	0.002	0.021		
$no_{\mathbf{x}}$	H	0.572	0.486	0.609	0.612	0.570		
	H, H <sup>2</sup>	0.772	0.568	0.658	0.615	0.653		
	Improvement	0.200 <sup>a</sup>	0.082ª	0.049a	0.003	0.083		

TABLE 7. CORRELATION COMPARISON FOR HUMIDITY

Temperature was not as strongly correlated with emissions as humidity, and this weaker correlation is indicated by the coefficients of determination in Table 8. Temperature had higher correlations with CO and HC than with NO<sub>X</sub>, particularly for the International-Perkins. Utilizing a quadratic fit, the average  $\rm r^2$  was 0.224 for CO, 0.176 for HC and 0.075 for NO<sub>X</sub>. The addition of the T<sup>2</sup> term to the linear fit of T resulted in an average increase in  $\rm r^2$  of 0.074 for HC, but only 0.018 for CO and 0.015 for NO<sub>X</sub>. Significant increases (p < 0.05) in  $\rm r^2$  occurred only when adding the T<sup>2</sup> term to the linear fit of T to HC for the Datsun and Mercedes.

a indicates improvement was significant at the 0.05 level

Additional regression equations were generated to determine whether or not humidity and temperature (together) predicted emissions better than these same independent variables fit separately, as in Tables 7 and 8. Linear as

well as stepwise polynomial regressions were calculated for each vehicle and each pollutant. The forms of generalized equations utilized are given as follows:

(3) 
$$\hat{E} = b_0 + b_1 H + b_2 T$$

(4) 
$$\hat{E} = b_0 + b_1 H + b_2 T + b_{12} H T$$

and (5) 
$$\hat{E} = b_0 + b_1 H + b_2 T + b_{11} H^2 + b_{22} T^2$$

where  $\hat{E}$ ,  $b_0$ ,  $b_1$ , and  $b_{11}$  are as defined in equations (1) and (2) and

b<sub>2</sub> = regression coefficient for linear effect of T

 $b_{12}$  = regression coefficient for the interaction effect of H and T

b22 = regression coefficient for quadratic effect of T

H = humidity (independent) variable:

and T = temperature (independent) variable.

The coefficients obtained in using equations (3), (4), and (5) with H and T are contained in Appendix B. Note that equations (4) and (5) were formed from a stepwise regression procedure in which H and T were forced into the equation. Consequently, at times only one of the quadratic terms ( $H^2$  or  $T^2$ ) entered the equation (with H and T) due to a low tolerance level on the other quadratic term.

TABLE 8. CORRELATION COMPARISON FOR TEMPERATURE

		Coefficie	Coefficients of determination (r <sup>2</sup> ) by vehicle					
Dependent		Dataun	Int'l. Perkins	Mercedes	Dougoot	7		
Variable	Variable(s)	Datsun	Perkins	Mercedes	Peugeot	Average		
нс	т т, т <sup>2</sup>	0.046	0.235 0.295	0.012 0.115	0.113 0.165	0.102 0.176		
	Improvement	0.131 0.085 <sup>a</sup>	0.293	0.103	0.163	0.074		
CO	т т, т <sup>2</sup>	0.003 0.024	0.424 0.471	0.128 0.131	0.267 0.268	0.206 0.224		
	Improvement	0.021	0.047	0.003	0.001	0.018		
NO <sub>x</sub>	т т, т <sup>2</sup>	0.055 0.070	0.037 0.051	0.077 0.099	0.069 0.079	0.060 0.075		
	Improvement	0.015	0.014	0.022	0.010	0.015		

a indicates improvement was significant at the 0.05 level

Table 9 contains the correlation comparisons for equation (3) with equations (1) and (2), and for equations (4) and (5) with equation (3). Coefficients of determination  $(r^2)$  are again used as the criterion for determining the best fits. The average  $r^2$  values for a fit linear in H and T were 0.194 for HC, 0.342 for CO, and 0.594 for NO<sub>X</sub>. For a fit including the cross product term (HT), the average  $r^2$  values were 0.254 for HC, 0.365 for CO, and 0.671 for NO<sub>X</sub>. For the quadratic fit, they were 0.272 for HC, 0.378 for CO, and 0.673 for NO<sub>X</sub>. Except for CO, the linear fit in H and T showed no significant improvement over the linear fit in H. This same fit was significantly (p < 0.05) better than the linear fit in T for NO<sub>X</sub> on all four vehicles; for CO on the Peugeot; and for CO and HC on the International-Perkins. The results thus indicate the strong relationship that exists between NO<sub>X</sub> and humidity for all Vehicles and between all emissions and ambient data for the International-Perkins.

TABLE 9. CORRELATION COMPARISON FOR HUMIDITY AND TEMPERATURE

<del></del>		Coefficient of determination (r <sup>2</sup> ) by vehicle				
Dependent	Independent		Int'l			Average
Variable	Variable	Datsun	Perkins	Mercedes	Peugeot	r <sup>2</sup>
<del> </del>						
HC	Н,Т	0.047	0.588	0.013	0.128	0.194
	Improv. over H	0.027	0.012	0.013	0.050	0.026
	Improv. over T	0.001	0.353 <sup>a</sup>	0.001	0.015	0.093
	. *		~			_
CO	H,T	0.090	0.764	0.144	0.372	0.342
	Improv. over H	0.040	0.067ª	0.142 <sup>a</sup>	0.372 <sup>a</sup>	0.155
	Improv. over T	0.066	0.340 <sup>a</sup>	0.016	0.105 <sup>a</sup>	0.132
$\mathbf{no}^{\mathbf{x}}$	H,T	0.601	0.523	0.614	0.638	0.594
	Improv. over H	0.029	0.037	0.005	0.026	0.024
	Improv. over T	0.546 <sup>a</sup>	0.486 <sup>a</sup>	0.537 <sup>a</sup>	0.569 <sup>a</sup>	0.535
нс	H,T, HT	0.063	0.591	0.031	0.330	0.254
пС	Improv. over H,T	0.063	0.003	0.031	0.330	0.234
	Improv. Over n,1	0.016	0.003	0.018	0.201	0.000
co	H,T, HT	0.095	0.781	0.207	0.378	0.365
00	Improv. over H,T	0.005	0.017	0.063	0.006	0.023
						1
$NO_{\mathbf{x}}$	H,T HT	0.717	0.601	0.713	0.654	0.671
^	Improv. over H,T	0.116	0.078	0.099	0.016	0.077
	0 24					
HC	н,т,н <sup>2</sup> ,т <sup>2b</sup>	0.153	0.621	0.119	0.194	0.272
	Improv. over H,T	0.106ª	0.033	0.106 <sup>a</sup>	0.066	0.078
	H,T,H <sup>2</sup> ,T <sup>2b</sup>		. 703	0.040	0 276	0.370
CO	1 ' ' '	0.102	0.793	0.242	0.376	0.378
	Improv. over H,T	0.012	0.029 <sup>a</sup>	0.098 <sup>a</sup>	0.004	0.036
NO	H,T,H <sup>2</sup> ,T <sup>2</sup> b	0.787	0.604	0.659	0.643	0.673
$NO^{\mathbf{X}}$	Improv. over H,T	0.787 0.186 <sup>a</sup>	0.004 0.081 <sup>a</sup>	0.035 0.045 <sup>a</sup>	0.005	0.079
<del></del>	Improv. Over H,T	0.100	L 0.001	0.043	1 0.003	0.075

a indicates significant improvement at 0.05 level

b H and T forced into equation, others entered by significance

Including the interaction term (HT) with the linear fit in H and T yielded an average increase in  $\rm r^2$  of 0.060 for HC, 0.023 for CO, and 0.077 for  $\rm NO_{\rm X}$ . These increases were significant (p < 0.05) for HC from the Peugeot and for  $\rm NO_{\rm X}$  from the other three vehicles.

The use of a quadratic equation in H and T instead of a linear fit yielded an average increase in  $r^2$  of 0.078 for HC, 0.036 for CO, and 0.079 for NO $_{\rm X}$ . These increases were significant (p < 0.05) for all emissions from the Mercedes, for CO and NO $_{\rm X}$  from the International-Perkins, and for HC and NO $_{\rm X}$  from the Datsun. For the NO $_{\rm X}$  emissions, the H $^2$  term was the main variable influencing the significant increases.

With the exception of results for the International-Perkins, low correlations existed between CO or HC and each of the independent variables in Tables 7, 8, and 9. Both CO and HC emissions from the International-Perkins were much more sensitive to humidity and temperature than expected, and no reason is known for this anomaly. In terms of engine design, the Perkins is not considered to be so grossly different than the others as to cause such results. One statistic which does set the International-Perkins apart from the other vehicles, however, is its weight-to-power ratio (all vehicles assumed loaded as light-duty vehicles). This value, in kg/kW, is 21.8 for the International-Perkins, 29.7 for the Datsun, 32.3 for the Mercedes, and 28.9 for the Peugeot. It is not known if the weight/power statistic is related to the anomaly noted above, but it does indicate that the Perkins engine was probably operating at a lower fraction of available power than the other engines (i.e., at lower F/A). In summary, due to the overall relatively low correlations associated with HC or CO and the ambient data, and because of the unusual results of these emissions in the International-Perkins vehicle, HC and CO corrections will not be discussed further in the text. The overall results from Tables 7, 8, and 9 confirm the existence of strong and consistent associations between NO, and humidity for all four vehicles, and it is this relationship that will be explored. Equations with H2 and T2 terms have also been compared to those with an HT interaction term, indicating that the quadratic form is a better predictor.

To examine the  $\mathrm{NO}_{\mathrm{X}}$ -humidity relationships in more detail, scatter plots have been constructed for each vehicle and are presented as Figures 12 through 15. The linear and quadratic equations in H have been plotted for each vehicle, and the coefficients of these equations are contained in Table 10. On the average, addition of the  $\mathrm{H}^2$  term increased  $\mathrm{r}^2$  from 0.570 to 0.653, which is a substantial improvement. Most of this improvement was made in fitting the curve for the Datsun 220C, which seemed to have a strong tendency toward "leveling off" of  $\mathrm{NO}_{\mathrm{X}}$  emissions as humidity increased above about 120 grains  $\mathrm{H}_2\mathrm{O}/\mathrm{1b}_{\mathrm{m}}$  dry air. Emissions of  $\mathrm{NO}_{\mathrm{X}}$  from the Datsun, International-Perkins, and Mercedes all tended to become less sensitive to humidity above some humidity level. Taking all four vehicles together, however, can still result in computation of a relationship which is useful for most ambient humidity values.

# COMPUTATION OF CORRECTION FACTORS

The regression equations generated in the previous subsection (RESULTS AND STATISTICAL ANALYSIS) were restricted to data on the individual vehicles,

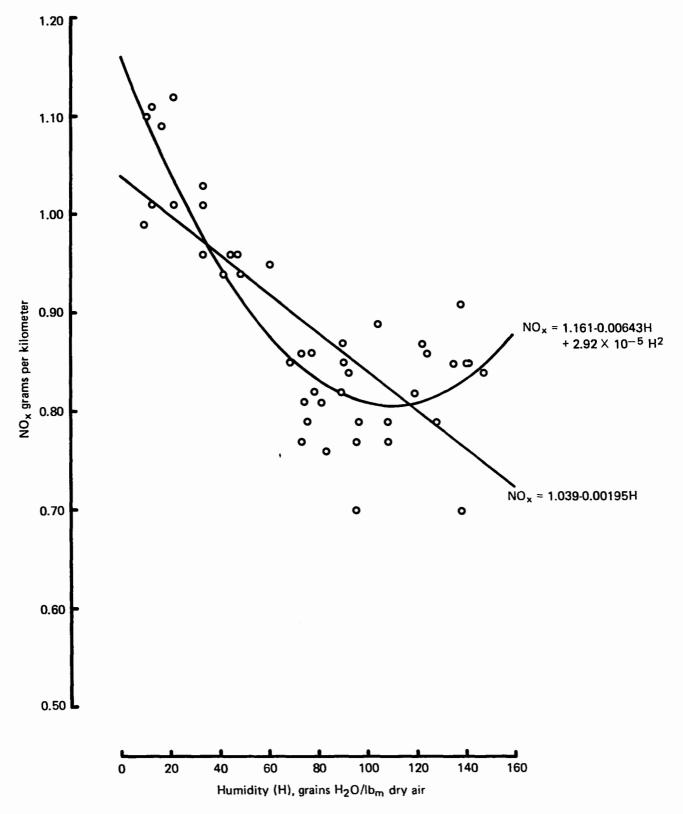


FIGURE 12. NO  $_{\rm X}$  EMISSIONS AS A FUNCTION OF HUMIDITY FOR A DATSUN 220C DIESEL SEDAN

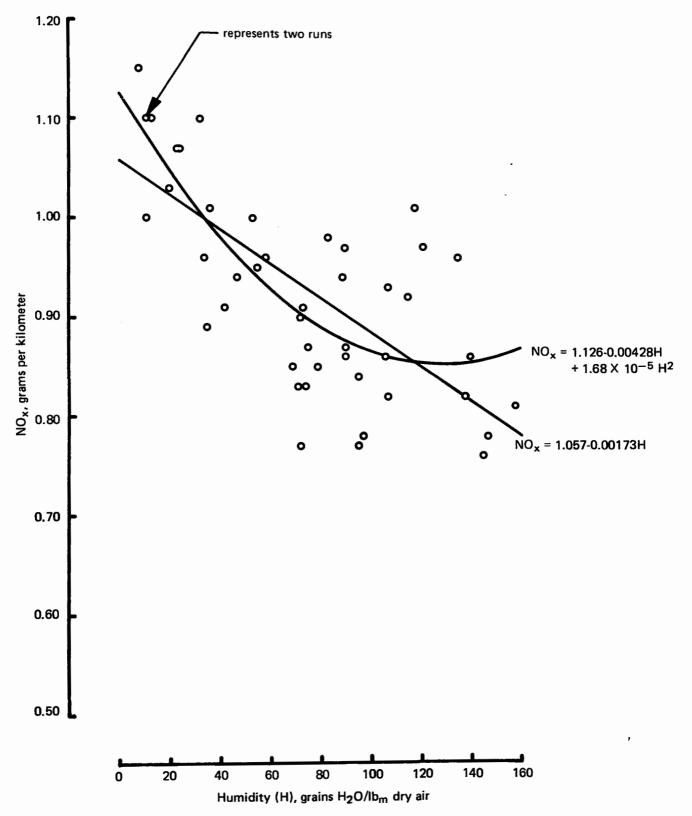


FIGURE 13. NO  $_{\rm X}$  EMISSIONS AS A FUNCTION OF HUMIDITY FOR AN INTERNATIONAL PICKUP TRUCK WITH PERKINS 6.247 DIESEL ENGINE

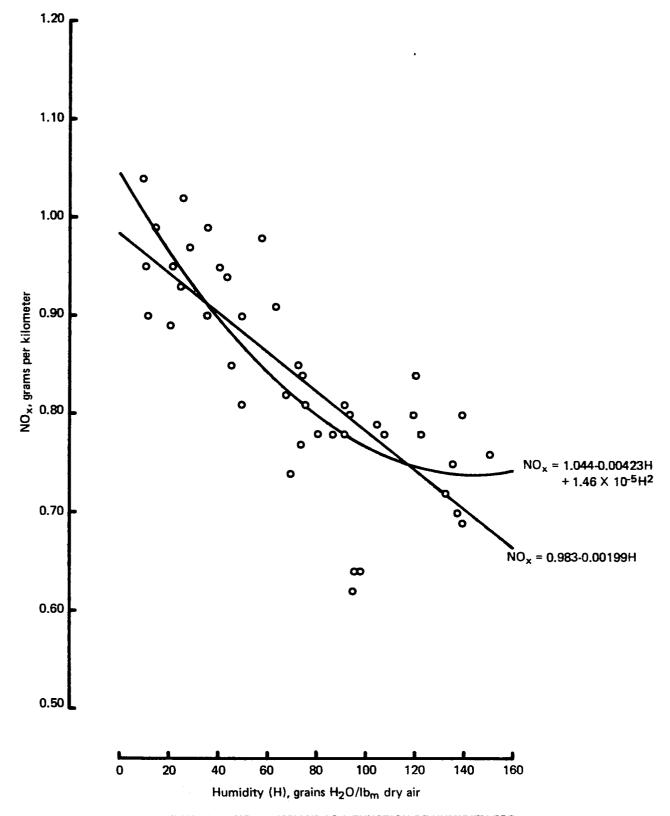


FIGURE 14. NO<sub>X</sub> EMISSIONS AS A FUNCTION OF HUMIDITY FOR A MERCEDES 240D DIESEL SEDAN

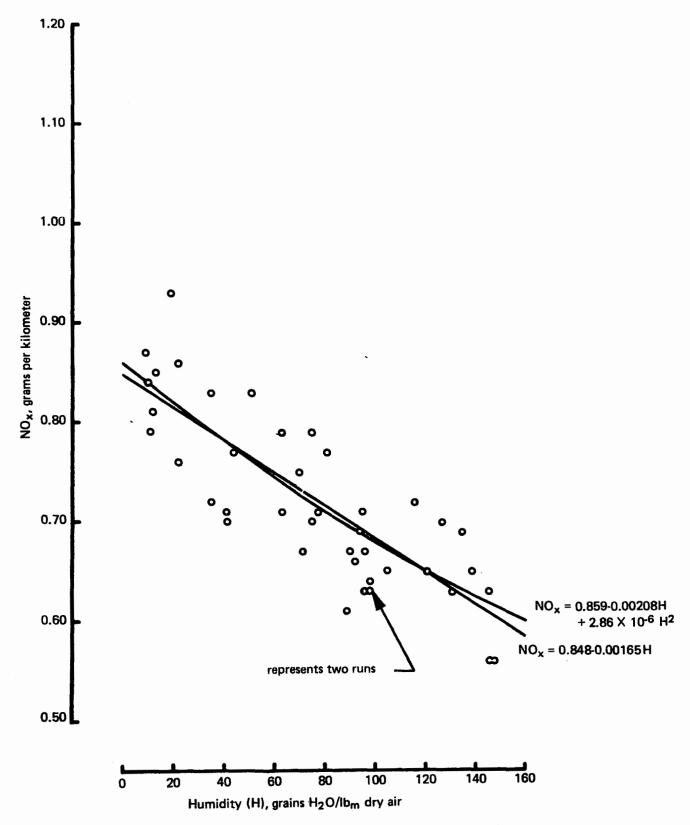


FIGURE 15.  $NO_X$  EMISSIONS AS A FUNCTION OF HUMIDITY FOR A PEUGEOT 504D DIESEL SEDAN

but the desired end result of this project is to obtain a single humidity correction factor for  ${\rm NO_X}$  applicable to the whole class of light-duty, diesel-powered vehicles.

Equation type <sup>a</sup>	Vehicle	(b <sub>O</sub> ) Constant	(b <sub>l</sub> ) H coefficient	(b <sub>ll</sub> ) H <sup>2</sup> coefficient	r <sup>2</sup>
Linear	Datsun	1.039	-0.00195		0.572
	IH-Perkins	1.057	-0.00173		0.486
	Mercedes	0.983	-0.00199		0.609
	Peugeot	0.848	-0.00165		0.612
Quadratic	Datsun	1.161	-0.00643	2.92 x 10 <sup>-5</sup>	0.772
-	IH-Perkins	1.126	-0.00428	$1.68 \times 10^{-5}$	0.568
	Mercedes	1.044	-0.00423	$1.46 \times 10^{-5}$	0.658

TABLE 10. COEFFICIENTS OF  $NO_X$  VERSUS HUMIDITY EQUATIONS

0.859

This factor would have the form

Peugeot

$$K_{h} = \frac{\text{predicted NO}_{x} \text{ value at H} = 75}{\text{predicted NO}_{x} \text{ value at H}}$$

where

linear case: predicted  $NO_X = b_0 + b_1 H = (b_0 + 75b_1) + b_1 (H-75)$ 

and

quadratic case: predicted 
$$NO_X = b_0 + b_1 H + b_{11} H^2$$
  
=  $(b_0 + 75b_1 + 75^2 b_{11}) + (b_1 + 150b_{11}) (H-75) + b_{11} (H-75)^2$ .

-0.00208

 $2.86 \times 10^{-6}$ 

0.615

Therefore, for the linear case,

$$K_h = \frac{1}{1 + L (H-75)}$$

where 
$$L = \frac{b_1}{b_0 + 75b_1}$$
;

and for the quadratic case,

$$K_h = \frac{1}{1 + L(H-75) + O(H-75)^2}$$

<sup>&</sup>lt;sup>a</sup> linear form  $NO_x = b_O + b_1 H$ , quadratic form  $NO_x = b_O + b_1 H + b_{11} H^2$ 

where 
$$L = \frac{b_1 + 2b_{11} (75)}{b_0 + b_1 (75) + b_{11} (75)^2}$$
 and 
$$Q = \frac{b_2}{b_0 + b_1 (75) + b_{11} (75)^2}.$$

Several methods were considered for combining data from the four vehicles to obtain the correction factors, including:

- 1. Obtaining a humidity correction factor for each vehicle separately and then averaging the results, i.e., the regression equations for the four vehicles would have different intercepts and difference slopes.
- 2. Establishing statistically a commonality of "H" coefficients in the linear  $NO_X$ -humidity equations, i.e., determining a common-slope, different-intercept model for the combined four vehicles.
- 3. Treating all data as if it were for a single vehicle, i.e., ignoring any vehicle-to-vehicle differences in magnitude of  $NO_X$  emissions and generating a common slope and common intercept model.

The first method, unequal slopes and intercepts, yielded four regression equations relating  $\mathrm{NO}_{\mathrm{X}}$  to H with an average  $\mathrm{r}^2$  of 0.570, and four equations relating  $\mathrm{NO}_{\mathrm{X}}$  to H and H² with an average  $\mathrm{r}^2$  of 0.653. The resulting coefficients were presented in Table 10. Notice from Table 7 that for three of the four vehicles the quadratic fit in H is significantly better than the linear fit. Humidity correction factors for  $\mathrm{NO}_{\mathrm{X}}$  were calculated for each vehicle for both the linear and quadratic equations. The results are given in Table 11 along with the average values of L and Q that would be utilized in establishing an overall humidity correction factor.

TABLE 11. SEPARATE VEHICLE CORRECTION FACTORS<sup>a</sup>

Equation Type and coefficient(s)	Datsun	IH-Perkins	Mercedes	Peugeot	Average
Linear					
L	-0.00218	-0.00187	-0.00239	-0.00228	-0.00218
Quadratic					
L	-0.00244	-0.00197	-0.00251	-0.00230	-0.00230
Q	3.46 x 10 <sup>-5</sup>	1.87 x 10 <sup>-5</sup>	1.81 x 10 <sup>-5</sup>	$0.40 \times 10^{-5}$	1.88 x 10 <sup>-5</sup>

$$K_{h} = \frac{1}{1 + L (75)} \text{ in linear case}$$

$$K_{h} = \frac{1}{1 + L (H-75) + Q (H-75)^{2}} \text{ in quadratic case}$$

The second method, common slope and unequal intercepts, produces some useful results. It is better than the first method in that it requires only one regression equation, but it also necessitates the separate computation of four correction factors. The common slope, unequal intercept method shows that the slopes of the  $NO_X$ -humidity equations for the four vehicles are close enough together to be considered equivalent statistically. The hypothesis that the four slopes are unequal was tested by comparing the mean squared errors of the regression equations calculated with and without a parallel line assumption. The resultant F statistic was not significant at the 0.05 level, so the common slope model was not rejected. The combined regression equation, which had an  $r^2$  of 0.722, is given after Table 12.

TABLE 12. COMMON SLOPE EQUATION COEFFICIENTS AND VALUES OF L

Vehicle	Intercept	Slope	L
Datsun	1.029	0.00183	-0.00205
Int'l.—Perkins	1.064	0.00183	-0.00197
Mercedes	0.972	0.00183	-0.00219
Peugeot	0.862	0.00183	-0.00252
		Average	-0.00218

Equation:  $NO_x = b_0 + b_1 X_1 + b_1 H$ , i = 2, 3, 4

where  $x_i = \begin{cases} 0 & \text{if "i th" vehicle data not used} \\ 1 & \text{if "i th" vehicle date used} \end{cases}$ 

Note: The intercept of the Datsun was  $b_0$ ; for the IH-Perkins,  $b_0 + b_2$ ; for the Mercedes,  $b_0 + b_3$ ; and for the Peugeot,  $b_0 + b_4$ .

Since each equation has a different intercept, the values of L in the linear  $K_{\rm h}$  factors are different for each vehicle. The average value, -0.00218, is the same as the average value obtained from Method l. Due to the difficulty in employing this method on quadratics, quadratic factors were not obtained.

The third method, common slopes and common intercepts, consisted of a simple combination of all 174 data points into a single regression equation relating  $\mathrm{NO}_{\mathbf{x}}$  to humidity. This approach was rejected due to the scatter and poor correlations resulting from combination of all the raw data. Two variations on this technique were then tried in an attempt to eliminate the adverse effects of differing  $\mathrm{NO}_{\mathbf{x}}$  magnitudes among the test vehicles.

The first variation consisted of standardizing the observed NO $_{\rm X}$  values from each vehicle by subtracting the vehicle mean NO $_{\rm X}$  and dividing by the vehicle NO $_{\rm X}$  standard deviation as follows:

$$\texttt{standardized NO}_{\mathbf{x}} = \frac{\texttt{observed NO}_{\mathbf{x}} - \texttt{mean NO}_{\mathbf{x}}}{\texttt{standard deviation NO}_{\mathbf{x}}}$$

The standardized values for all four vehicles were then combined and fit to humidity in a single regression equation. The results for the linear and quadratic fits are given below:

standardized  $NO_{x} = 1.362 - 0.01794 H, r^{2} = 0.566$ 

standardized NO<sub>X</sub> = 1.976 - 0.04080 H + 0.000150 H<sup>2</sup>,  $r^2$  = 0.631.

There was a significant increase (p < 0.0001) in the goodness of fit utilizing the quadratic equation in H as compared to the linear model.

To obtain the humidity correction factors for  $\mathrm{NO}_{\mathbf{X}}$  (unstandardized), the values of L and Q were calculated, adjusting for the different means and standard deviations of each vehicle's data. These values are given in Table 13 along with their averages. The average results are again in good agreement with Methods 1 and 2.

Equation Type	Datsun	IH-Perkins	Mercedes	Peugeot	Average
Linear L	-0.00218	-0.00200	-0.00226	-0.00223	-0.00217
Quadratic L	-0.00229	-0.00210	-0.00239	-0.00235	-0.00228
Q	1.88 x 10 <sup>-5</sup>	1.72 x 10 <sup>-5</sup>	1.95 x 10 <sup>-5</sup>	1.92 x 10 <sup>-5</sup>	1.87 x 10 <sup>-5</sup>

TABLE 13. STANDARDIZED NOx CORRECTION FACTORS

The second variation on Method 3 consisted of computing the ratios of the observed NO $_{\rm X}$  emissions data to the predicted NO $_{\rm X}$  values at an arbitrary humidity point for each vehicle. These "normalized" data were then combined to derive a common regression equation. The equations given in Table 10, both linear and quadratic, were used to compute a predicted value for NO $_{\rm X}$  at H = 75 for each vehicle. Normalized NO $_{\rm X}$  values were then obtained (separately for quadratic and linear models) using the following definition:

normalized 
$$NO_X = \frac{\text{measured } NO_X}{\text{predicted } NO_X}$$
 at H = 75

Combined equations relating normalized  $NO_X$  to humidity were then generated, and the resulting coefficients are given in Table 14. The quadratic fit yielded a significant (p < 0.001) improvement in correlation over the linear equation, supporting similar findings from the other methods utilized in this project. The definition of normalized  $NO_X$  forces the normalized  $NO_X$  values calculated by both the linear and quadratic equations to be 1.0 at the value H = 75. This second variation, using normalized  $NO_X$  values, is the

only method evaluated which yields a single factor for correction of  ${\rm NO}_{\rm X}$  without averaging values of L and Q. The computed coefficients for the two cases are

L = -0.00217 for the linear case,

and L = -0.00228,  $Q = 1.85 \times 10^{-5}$  for the quadratic case.

TABLE 14. COEFFICIENTS OF NORMALIZED  $\mathrm{NO}_{\mathbf{x}}$  VERSUS HUMIDITY EQUATIONS, ALL VEHICLES COMBINED

Equation type <sup>a</sup>	(b <sub>O</sub> ) constant	(b1) H coefficient	(b <sub>11</sub> ) H <sup>2</sup> coefficient	r <sup>2</sup>
Linear	1.163	-0.00217		0.569
Quadratic	1.274	-0.00504	1.84 x 10 <sup>-5</sup>	0.616

a linear form, normalized  $NO_X = b_0 + b_1H$ ; quadratic form, normalized  $NO_X = b_0 + b_1H + b_{11}H^2$ 

These results are very similar to those obtained using the other methods, as shown in Table 15.

TABLE 15. SUMMARY OF HUMIDITY CORRECTION FACTOR RESULTS<sup>a</sup>

	Linea	r Case	Qu	adratic Case		
	Average	Average	Average	Average	Average	
Method	r <sup>2</sup>	L	r <sup>2</sup>	L	Q	
Unequal slopes and unequal	0.570	0.0010	0.652	0.00000	1 00 - 10=5	
intercepts	0.570	-0.00218	0.653	-0.00230	1.88 x 10 <sup>-5</sup>	
Equal slopes but unequal intercepts	0.722	-0.00218				
Equal slopes and equal intercepts						
a) Std. ${ m NO}_{ m x}$	0.566	-0.00217	0.631	-0.00228	1.87 x 10 <sup>-5</sup>	
b) Nor. NO <sub>x</sub>	0.569	-0.00217	0.616	-0.00228	1.85 x 10 <sup>-5</sup>	

a linear case  $K_h = \frac{1}{1 + L (H-75)}$ quadratic case  $K_h = \frac{1}{1 + L (H-75) + Q (H-75)^2}$ 

Correction factors representing the averages of the linear and quadratic results given in Table 15 are shown in Figure 16.

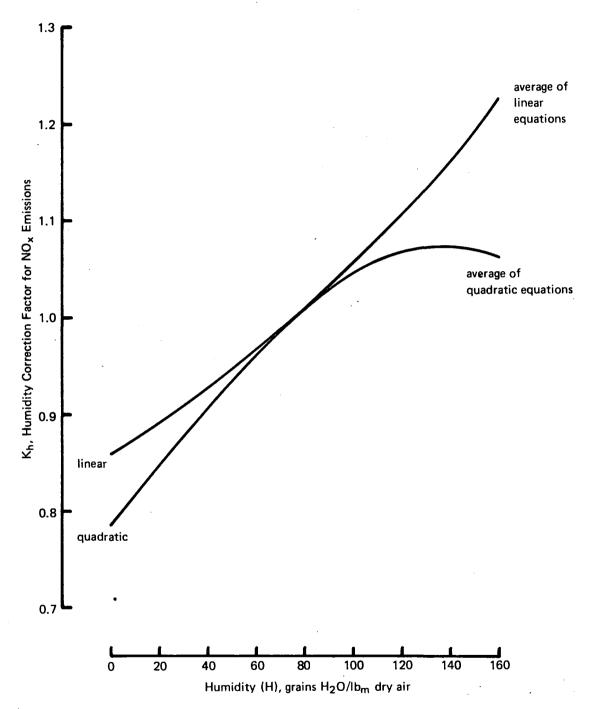


FIGURE 16. HUMIDITY CORRECTION FACTOR  $\rm K_h$  FOR NO  $\rm _x$  EMISSIONS AS A FUNCTION OF HUMIDITY, AVERAGE OF RESULTS GIVEN IN TABLE 15

#### SECTION 6

#### COMPARISON WITH OTHER CORRECTION FACTORS

Studies were conducted in the past on other classes of vehicles and engines with the aim of determining applicable factors for the correction of measured emissions to "standard" ambient conditions. The classes studied were light-duty, gasoline-fueled vehicles  $^{(5)}$ ; heavy-duty diesel engines  $^{(6)}$ ; and heavy-duty, gasoline-fueled vehicles. The results of these studies have appeared in corresponding Federal Emission Regulations  $^{(1,3,7,8)}$  as correction factors for NO<sub>X</sub> emissions, other ambient effects having been considered negligible by EPA.

The existing correction factors for NO, are:

light-duty gasoline; 
$$K_h = \frac{1}{1 - 0.0047 \text{ (H-75)}}$$

heavy-duty diesel; 
$$K_h = \frac{1}{1+(0.044 \text{ F/A}-0.0038) (H-75)+(-0.116 \text{ F/A}+0.0053) (T-85)}$$

heavy-duty gasoline;  $K_h = 0.634 + 0.00654H - 0.0000222H^2$ .

In addition, a simpler correction factor was used for heavy-duty diesels through about mid-1974; and it was

$$K_h = \frac{1}{1 - 0.0025 \text{ (H-75)}}$$
.

These factors can be compared to those generated by this project (results of equal slope/intercept-normalized  $NO_X$  method shown for example), which are:

based on linear equation; 
$$K_h = \frac{1}{1 - 0.00217 \text{ (H-75)}}$$

based on quadratic equation; 
$$K_h = \frac{1}{1 - 0.00228 \text{ (H-75)} + (1.85 \times 10^{-5}) \text{ (H-75)}^2}$$
.

The most striking comparison which can be made, of course, is that the light-duty diesel factor based on a linear NO<sub>X</sub>-humidity relationship is very similar to the original (and since replaced) factor for heavy-duty diesels. This light-duty diesel factor shows less sensitivity to humidity than the light-duty gasoline factor. These relationships are given in tabular form in Table 11 and in graphical form in Figure 17. The range shown in Figure 17 for the current heavy-duty diesel factor incorporates all the expected variation

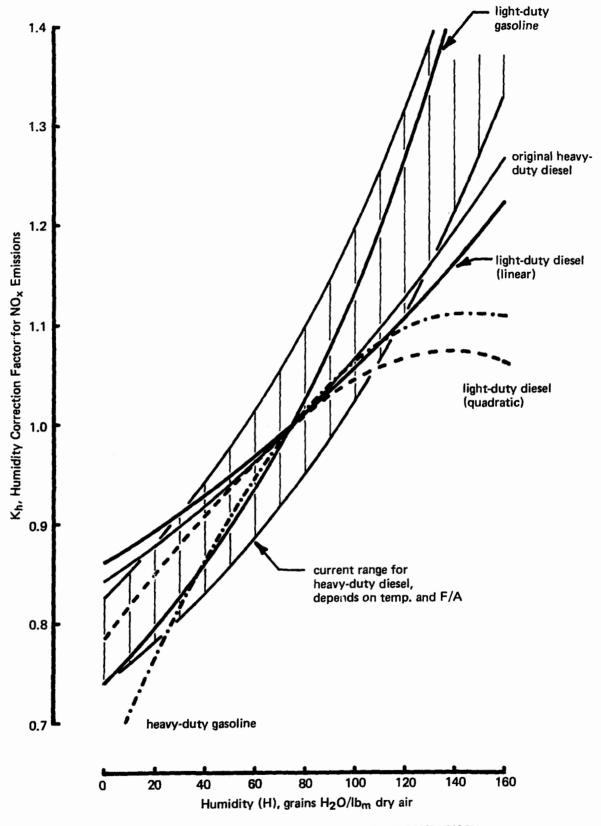


FIGURE 17. HUMIDITY CORRECTION FACTOR COMPARISON

in both test temperature (70°F to 100°F) and F/A ratio (0.005 to 0.07). This factor, by using F/A as a variable, is restricted to modal steady-state engine operation.

The heavy-duty gasoline factor and the quadratic-based light-duty diesel factor are qualitatively similar, with the gasoline factor being a stronger function of humidity. Application of either the linear-based or the quadratic-based factor computed from data acquired in this task would be a relatively simple matter, and it remains for the sponsor to decide whether or not the significantly better fit of the quadratic form is sufficient reason to deviate from the customary linear-based form for the light-duty diesel. Reasons for the differences between correction factors discussed here include not only type and size of engine, but also the difference between heavy- and light-duty gasoline correction factors).

TABLE 16. HUMIDITY CORRECTION FACTORS FOR  $NO_{\mathbf{x}}$  IN TABULAR FORM

midity (H), grains H <sub>2</sub> O/	Light-duty diesel		Heavy-duty die	sel(3,8)	Light-duty	Heavy-duty
lbm/dry air	Linear	Quadratic	Current, range	Original	gasoline(1)	gasoline <sup>(7)</sup>
0	0.860	0.785	0.738 to 0.825	0.842	0.739	0.634
20	0.893	0.847	0.782 to 0.880	0.879	0.795	0.756
40	0.929	0.907	0.831 to 0.942	0.920	0.859	0.860
60	0.968	0.963	0.887 to 1.014	0.964	0.934	0.946
80	1.011	1.011	0.951 to 1.099	1.013	1.024	1.015
100	1.057	1.048	1.024 to 1.198	1.067	1.133	1.066
120	1.108	1.070	1.110 to 1.317	1.127	1.268	1.099
140	1.164	1.076	1.212 to 1.463	1.194	1.440	1.114
160	1.226	1.065	1.334 to 1.645	1.270	1.665	1.112

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- 6. S. R. Krause, D. F. Merrion, and G. L. Green, "Effect of Inlet Air Humidity and Temperature on Diesel Exhaust Emissions." Paper 730213 presented at SAE Meeting, Detroit, January 8-12, 1973.
- 7. Federal Register, Volume 37, No. 221 (Subpart H), November 15, 1972.
- 8. Federal Register, Volume 38, No. 124, June 28, 1973.

APPENDIX A

TABULAR DATA

TABLE A-1. TABULAR DATA BY RUN

Run	Ţ								т	
	1	**-1-2-7		issions		·		gr H <sub>2</sub> 0/	Temp.,	ра,
No.	Date	Vehicle	HC	CO	NOX	CO <sub>2</sub>	km/l	lb <sub>m</sub> āir	°F	in Hg
1	12/12/75	I.HPerkins	0.47	,	0 00					
2	1	Mercedes	0.47	1.95	0.90	244.86	10.8	72.4	76.3	29.18
3	12/12/75	\$	0.18	0.63	0.77	238.15	11.2	74.1	77.6	29.08
4	1	Mercedes	0.17	0.80	0.81	222.85	12.0	74.9	78.4	29.08
5	12/15/75		0.13	0.63	0.90	248.44	10.8	50.4	73.1	29.28
,	12/13/73	Datsun	0.16	0.75	0.96	231.17	11.5	44.6	72.5	29.37
6	12/15/75	Peugeot	0.34	1.01	0.70	229.47	11.6	40.9	71.2	29.37
7	12/15/75	I.HPerkins	0.62	1.46	0.91	232.97	11.3	41.6	70.3	29.35
8	12/16/75	Mercedes	0.14	0.65	0.99	261.35	10.2	35.8	75.8	29.28
9	12/16/75	Datsun	0.12	0.76	0.96	231.24	11.5	34.3	75.2	29.28
10	12/16/75	Peugeot	0.44	1.03	0.72	234.61	11.3	35.2	77.8	29.27
11		I.HPerkins	0.59	2.01	0.96	247.37	10.7	34.3	77.1	29.24
12		Mercedes	0.11	0.61	0.85	235.48	11.4	45.9	70.2	29.28
13	12/17/75	h	0.08	0.77	0.94	228.91	11.7	42.4	67.3	29.29
14	12/17/75		0.42	1.09	0.71	232.56	11.4	41.3	68.0	29.30
15	12/17/75	I.HPerkins	0.63	1.75	0.89	219.50	12.0	35.1	68.0	29.27
16	12/18/75	Mercedes	0.15	0.61	0.90	242.40	11.0	11.6	76.9	29.78
17	12/18/75	1	0.14	0.69	0.99	218.01	12.2	10.1	76.4	29.78
18	12/18/75		0.37	0.95	0.84	230.91	11.5	10.1	76.0	29.78
19	12/18/75	· -	0.58	1.71	1.00	232.55	11.3	10.4	79.0	29.78
20								•		•
20	1/7/76	Mercedes	0.10	0.69	0.99	255.01	10.5	15.0	67.1	29.40
21	1/7/76	Datsun	0.15	0.74	1.11	250.82	10.6	13.2	67.1	29.40
22	1/7/76	Peugeot	0.39	1.01	0.85	255.42	10.4	13.0	67.2	29.39
23	1/7/76	I.HPerkins	0.54	1.91	1.10	258.16	10.2	12.9	67.4	29.39
24	1/8/76	I.HPerkins	0.54	1.96	1.15	258.77	10.2	8.3	66.8	29.69
25	1/8/76	Peugeot	0.34	1.10	0.87	257.67	10.3	9.1	67.4	29.69
	, ,	_				1				
26	1/8/76	Datsun			Discar			<u> </u>		
27	1/8/76	Mercedes	0.15	0.67	1.04	237.25	11.3	9.8	68.4	29.60
28	1/9/76	Mercedes		- Data	Discar	ded ——		<del></del>		
29	1/9/76	Datsun	0.19	0.83	1.09	233.43	11.4	17.1	77.8	29.51
30	1/9/76	I.HPerkins	0.49	1.87	1.03	245.01	10.8	20.1	76.9	29.48
	10 /74	D t	Ĺ	Data	Diasan	 			į	
31	1/9/76	Peugeot	2 50		Discar	246.72	10.7	72.2	77.7	29.09
	1/13/76	I.HPerkins	0.59	2.12	3		11.2	73.7	77.4	29.14
33	1/13/76	Datsun	0.18	0.84	0.77	239.08	i e	70.8	77.2	29.14
34	1/13/76	Peugeot	0.51	1.00	0.67	240.72	11.0	1		4
35	1/13/76	Mercedes	0.23	0.70	0.91	256.44	10.4	63,9	77.2	29.09
36	1/14/76	Mercedes	0.17	0.69	0.93	256.81	10.4	24.6	68.1	29.60
37	1/14/76	I.HPerkins	0.55	1.94	1.07	259.60	10.2	24.0	67.7	29.60
38	1/14/76	Peugeot	0.39	1.03	0.76	235.07	11.3	21.8	68.0	29.60
39	1/14/76	Datsun	0.15	0.86	1.01	242.44	11.0	21.5	67.6	29.50
40	1/16/76	Datsun	0.07	0.81	1.03	241.13	11.1	33.9	77.2	29.36
		L	Ļ <u>.</u>		<u> </u>	<u></u>	<u></u>	<del></del>	<u> </u>	<del></del>

TABLE A-1 (continued)

	<del></del>	r							·	
Run				ssions				gr H <sub>2</sub> O/	Temp.,	pa,
No.	Date	Vehicle	HC	co	NOx	CO <sub>2</sub> .	km/l	$lb_{m}$ air	°F	in Hg
41	1/16/76	Wa	0.13	0.60	1 00	261 14	10.0	26.2	77.0	20. 20
		Mercedes	0.13	0.69	1.02	261.14	10.2	26.3	77.2	29.38
42	1/16/76	I.HPerkins	0.50	2.04	1.07	252.37		23.2	76.8	29.38
43	1/16/76	Peugeot	0.42	0.99	0.86	237.09	1	21.6	77.6	29.38
44	1/19/76	Datsun	0.14	0.83	0.96	250.13	r .	47.8	86.6	29.35
45	1/19/76	I.HPerkins	0.50	2.13	0.94	260.42	10.1	47.4	86.6	29.35
46	1/19/76	Mercedes	0.15	0.66	0.94	266.22	10.0	44.5	86.7	29.35
47	1/19/76	Peugeot	0.39	1.04	0.77	249.75	10.6	i	86.2	29.34
48	1/20/76	I.HPerkins	0.52	1.99	1.10	273.05		1	76.9	29.63
49	1/20/76	Mercedes	0.12	0.66	0.97	257.48	10.4	t .	77.7	29.65
50	1/20/76	Datsun	0.12	0.82	1.12	252.65	10.6	21.9	78.1	29.63
51	1/20/76	Peugeot	0.40	1.07	0.93	257.28		t -	76.9	29.59
52	1/21/76	Datsun	0.16	0.80	1.01	228.68		ľ	86.5	29.65
53	1/21/76	Peugeot	0.43	0.95	0.79	233.46		l .	86.7	29.60
54	1/21/76	I.HPerkins	0.51	1.98	1.10	•	10.2	10.6	86.3	29.56
55	1/21/76	Mercedes	<del></del>	Data	Discar	ded				
	<b>]</b>									
56	1/22/76	Peugeot	0.48	0.88	0.81	227.68	11.7		87.7	29.53
57	1/22/76	Datsun	0.07	0.77	1.10	229.04		11.1	88.1	29.65
58	1/22/76	I.HPerkins	0.59	1.93	1.10	266.63	9.9	11.2	86.0	29.48
59	1/22/76	Mercedes	0.12	0.61	0.95	238.41	11.2	10.9	87.6	29.41
60	1/23/76	Mercedes	0.13	0.59	0.95	247.04	10.8	22.3	87.3	29.20
61	2/11/76	Daugast	0.44	1.06	0.67	253 06	10.5	00.6	77.3	20.44
62		Peugeot	ł			251.86		89.6	1 !	29.44
	2/11/76	Datsun	0.15	0.83	0.86	259.23	10.3	78.4	78.1	29.43
63	2/11/76	I.HPerkins	0.60	2.11	0.91	267.66	9.9	1	76.3	29.40
64	2/11/76	Mercedes	0.15	0.69	0.84	256.56		74.8	78.3	29.38
65	2/13/76	Peugeot	0.48	1.04	0.71	244.35	10.9	62.8	68.9	29.27
66	2/13/76	I.HPerkins	0.60	2.13	0.87	264.66	10.0	75.4	77.9	29.28
67	2/13/76	Datsun	0.13	0.79	0.85	235.51	11.3	69.1	76.7	29.28
68	2/13/76	Mercedes	0.14	0.64	0.82	250.85			78.2	29.28
69	2/16/76	Datsun	0.12			236.62			78.7	29.08
70	2/16/76	Mercedes	<b> </b>		Discar			<b></b>		
			]		1				1	
71	2/16/76	I.HPerkins	0.63	2.20	0.78	260.02	10.1	96.6	78.7	29.29
72	2/16/76	Peugeot	0.42	1.02	0.63	232.27	11.4	98.1	78.7	29.25
73	2/17/76	Datsun	0.16	0.92	0.79	253.55	10.5		87.9	28.98
74	2/17/76	I.HPerkins	0.75	2.79	0.82	279.71	9.4	107.4	86.8	29.03
75	2/17/76	Peugeot	0.35	1.04	0.79	247.52		1	85.3	28.94
				}	}			]		
76	2/17/76	Mercedes	0.10	0.67	0.95	265.76	i .	41.1	87.2	28.88
77	2/26/76	I.HPerkins	0.58	2.16	0.95		,	,	70.9	29.37
78	2/26/76	Mercedes	0.08	0.68	0.98		10.8	57.7	69.0	29.35
79	2/26/76	Datsun		Date	a Disca	rded		<del> </del>		
80	2/26/76	Peugeot		Data	a Disca	rded —				
	L		L	L	L	L	L	L	L	<u> </u>

TABLE A-1 (continued)

Run	Γ		Pont	i and an a		/>	The . 1		I =	
No.	Date	Vehicle	HC	co CO	NO <sub>X</sub>	CO <sub>2</sub>	Fuel, km/l	gr H <sub>2</sub> O/	Temp.,	pa,
	Duce	Venicle	IIC .		NOX	CO2	KIII/ X	lb <sub>m</sub> air	P.	in Hg
81	2/27/76	I.HPerkins	0.65	1.97	0.96	256.82	10.3	58.2	68.9	29.35
82	2/27/76	Mercedes			Discar			30.2	00.3	23.33
83	2/27/76	Datsun	0.20	0.87		236.82	11.3	61.4	68.3	29.35
84	2/27/76	Peugeot			Discar					
85	3/1/76	Mercedes	0.14	0.64	0.78	238.18	11.2	92.0	71.1	29.03
86	3/1/76	Datsun	0.13	0.92	0.85	229.33	11.6	91.3	70.6	29.03
<b>87</b>	3/1/76	I.HPerkins	0.71	2.52	0.87	251.94	10.4	90.3	70.1	29.02
88	3/1/76	Peugeot	0.49	1.19	0,66	241.70	11.0	92.0	70.9	29.01
89	3/2/76	Datsun	0.14	0.83	0.79	231.78	,	96.8	71.0	29.05
90	3/2/76	Peugeot	0.41	1.09	0.63	238.19	11.1	95.7	70.7	29.07
91	3/2/76	I.HPerkins	0.68	2.29	0.84	254.02		95.0	70.2	29.05
92	3/2/76.	Mercedes	0.11	0.61	0.64	212.53	_	97.6	70.6	29.04
93	3/3/76	Peugeot	0.47	1.03	0.64	231.95		98.3	76.5	29.06
94	3/3/76	Datsun	0.15	0.79	0.77	228.34		95.9	76.1	29.04
95	3/3/76	I.HPerkins	0.67	2.07	0.77	252.52	10.4	94.6	76.3	29.04
06	2 /2 /76	Wanna 3	0.00	0.62	0.64	233.37	11.5	96.0	76.7	29.02
96 97	3/3/76	Mercedes	0.08 0.11	0.62	0.62	233.37		95.0	76.8	28.94
97 98	3/4/76 3/4/76	Mercedes Peugeot	0.11	1.06	0.62	251.63		88.6	79.1	28.80
99	3/4/76	Datsun	0.20	0.84	0.76	236.91	11.3	83.6	78.3	28.87
100	3/4/76	I.HPerkins	0.72	1.97	0.83	273.91	9.6	74.5	77.3	28.83
100	3/4/10	1.II. TELKIIIS	0.72	2.57	0.00	-/017-				
101	3/8/76	Datsun	0.15	0.86	0.79	230.62	11.6	75.9	69.4	29.07
102	3/11/76	I.HPerkins	0.64	1.82	0.85	236.79		69.4	68.4	29.14
103	3/11/76	Mercedes	0.12	0.67	0.85	259.20	10.3	73.3	68.3	29.13
104	3/11/76	Peugeot	0.40	1.03	0.70	238.32	11.1	75.4	68.0	29.09
105	3/12/76	Peugeot	0.49	1.03	0.63	230.91	11.5	100.8	77.1	28.96
									ł	
106	3/12/76	Mercedes	0.39	0.66	0.79	251.34	1	104.7	77.8	28.99
107	3/12/76	I.HPerkins	0.77	2.27	0.86	255.72		105.9	75.1	28.98
108	3/12/76	Datsun	0.18	0.71	0.77	224.62		108.7	75.4	28.99
109	3/19/76	Datsun	0.21	0.80	0.82	246.41	B.	78.6	85.7	29.02
110	3/19/76	Peugeot	0.30	0.94	0.79	247.13	10.8	75.1	85.2	29.02
	ł						١		05.0	
111	3/19/76	I.HPerkins	0.71	2.12	0.83	238.37	1	71.2	85.8	29.02
112	3/19/76	Mercedes	0.13	0.60	0.74	238.35	1	70.1	86.1	28.96
113	3/22/76	Peugeot	0.36	0.92	0.83	229.84	Į.	34.7	76.8	29.41 29.43
114	3/22/76	Datsun	0.19	0.77	1.01	238.24	•	33.9	77.3	29.43
115	3/22/76	Mercedes	0.12	0.59	0.90	236.94	11.3	36.4	86.5	27.41
			2.75	2 14	, ,,	252 06	10.4	36.3	87.3	29.37
116	3/22/76	I.HPerkins	0.75	2.14	1.01	253.96 227.24	1	50.2	75.4	29.44
117	3/23/76	Mercedes	0.12	0.59 0.83	0.81	238.23		50.6	75.8	29.42
118	3/23/76	Peugeot	0.29 0.65	2.15	1.00	258.82		52.7	77.7	29.43
119	3/23/76	I.HPerkins	0.65	0.79	0.94	235.13		49.2	77.3	29.43
120	3/23/76	Datsun	0.23	0.75	V. J.					

TABLE A-1 (continued)

		r	Em			73				<del></del>
Run	<b>D</b> -4-	77-1-2-1		issions,			Fuel,	gr H <sub>2</sub> O/	Temp.,	pa,
No.	Date	Vehicle	HC	CO	NOX	CO <sub>2</sub>	km/火	lb <sub>m</sub> air	· F	in Hg
121	3/24/76	Peugeot	0.38	0.86	0.75	240.96	11.0	69.9	87.1	29.19
122	3/24/76	Mercedes	0.15	0.63	0.73	252.13		76.3	86.9	29.22
123	3/24/76	I.HPerkins	0.75	2.49	0.85	254.38		78.7	86.8	29.09
124	3/24/76	Datsun	0.14	0.81	0.86	242.65	11.0	75.7	86.4	29.09
125	3/29/76	Datsun	0.14	0.80	0.79	242.65	10.8	128.8	86.8	28.96
123	3/23/10	Dacsui	0.10	0.80	0.79	240.31	10.0	120.0	80.0	20.90
126	3/29/76	Peugeot	0.40	0.97	0.69	233.01	11.4	94.3	87.0	28.94
127	3/29/76	Mercedes	0.11	0.62	0.89	249.12	10.7	21.1	86.9	28.85
128	4/6/76	I.HPerkins	0.67	2.27	0.98	263.84	10.0	83.1	87.8	29.12
129	4/6/76	Datsun	0.11	0.65	0.81	226.82	11.8	82.4	87.1	29.12
130	4/6/76	Peugeot	0.34	0.95	0.77	239.27	11.1	80.8	85.8	29.10
131	4/6/76	Mercedes	0.15	0.50	0.78	227.21	11.8	81.3	87.4	29.08
132	4/14/76	I.HPerkins	0.66	2.59	0.97	278.80	9.4	120.7	86.8	29.15
133	4/14/76	Mercedes	0.13	0.59	0.78	253.08	10.6	122.7	86.7	29.15
134	4/14/76	Datsun	0.12	0.80	0.87	256.75	10.4	122.6	86.2	29.15
135	4/14/76	Peugeot	0.30	1.00	0.65	226.82	11.7	121.1	86.4	29.14
								1		
136	5/5/76	I.HPerkins	0.69	2.41	1.01	274.62		118.1	86.4	29.08
137	5/5/76	Mercedes	0.14	0.63	0.84	270.13	9.9	120.6	87.1	29.09
138	5/5/76	Datsun	0.04	0.79	0.86	250.30	10.7	124.7	87.2	29.09
139	5/5/76	Peugeot	0.31	0.92	0.70	235.26	1	127.4	86.0	29.04
140	5/6/76	Mercedes	0.13	0.61	0.80	268.73	10.0	119.7	87.3	29.08
141	5/6/76	Datsun	0.09	0.77	0.82	253.11	10.6	120.0	87.5	29.09
142	5/6/76	Peugeot	0.34	0.94	0.72	238.22	1	115.7	86.3	29.11
143	5/6/76	I.HPerkins	0.76	2.55	0.92	270.55	t	114.6	86.8	29.12
144	5/25/76	Datsun	0.15	0.86	0.91	272.27		139.5	87.7	28.99
145	5/25/76	Mercedes	0.10	0.71	0.80	268.54	10.0	139.6	88.0	28.97
146	5/25/76	I.HPerkins	0.83	2.83	0.96	286.52	9.2	135.2	87.0	28.95
147	5/25/76	Peugeot	0.26	0.98	0.69	246.39	Į.	135.4	88.1	28.91
148	5/27/76	Datsun	0.17	0.76	0.89	248.63	l	1	86.1	29.21
149	5/27/76	I.HPerkins	0.76	2.40	0.93	271.77	ľ	1	86.9	29.21
150	5/27/76	Mercedes	0.11	0.61	0.78	267.83		108.5	87.4	29.21
130	3/2///6	Mercedes	0.11	0.01	0.76	207.03	10.0	108.3	07.4	29.21
151	5/27/76	Peugeot	0.25	0.91	0.65	225.99	11.8	105.1	86.6	29.23
152	5/28/76	I.HPerkins	0.62	2.31	0.97	260.27		90.3	84.8	29.17
153	5/28/76	Mercedes	0.13	0.62	0.81	248.16		· ·	87.4	29.18
154	5/28/76	Datsun	0.06	0.74	0.87	240.38	1	90.7	87.0	29.16
155	5/28/76	Peugeot	0.33	0.94	0.71	235.80	1	95.2	88.2	29.14
133	3/20/10	reageot	0.33	0.54	0.71	233.00	11.5	33.2	00.2	27.14
156	5/31/76	Datsun	0.09	0.80	0.85	243.03	11.0	142.2	87.1	28.99
157	5/31/76	Peugeot	0.09	1.00	0.63	243.00		1	87.1	28.97
158	5/31/76	I.HPerkins	0.64	2.60	0.78	254.23			88.6	28.95
159	5/31/76	Mercedes	0.05	0.63	0.72	253.24	I .	132.8	87.3	28.94
160	6/1/76	Peugeot	0.28	0.98	0.65	237.69	ı	1	85.7	29.00
	-/-//	1	1		1				1 22	

TABLE A-1 (continued)

Run			Em	issions	, grams	/km	Fuel,	gr H <sub>2</sub> O/	Temp.,	pa,
No.	Date	Vehicle	HC	CO	иох	co <sub>2</sub>	km/l	lb <sub>m</sub> áir	°F	in Hg
161	6 /3 /36	D-1	0.10	0.00	0.05					
161	6/1/76	Datsun	0.12	0.83	0.85	248.71	10.7	136.5	85.0	29.09
162	6/1/76	I.HPerkins	0.79	2.77	0.82	269.05	9.7	137.6	86.1	29.16
163	6/1/76	Mercedes	0.14	0.61	0.69	252.23	10.6	140.0	86.6	29.15
164	6/2/76	Peugeot	0.47	1.01	0.56	252.85		146.5	87.8	29.14
165	6/2/76	Datsun	0.12	0.83	0.70	262.95	10.2	138.9	85.9	29.16
166	6 /2 /76									
166	6/2/76	I.HPerkins	1.00	3.03	0.76	270.22	9.7	134.9	86.4	29.16
167	6/2/76	Mercedes	0.10	0.67	0.70	268.86	1	137.9	86.5	29.16
168	6/3/76	Peugeot	0.35	1.00	0.67	240.36		95.9	87.4	29.15
169	6/3/76	I.HPerkins	0.80	2.54	0.86	264.11	9.9	90.4	87.4	29.15
170	6/3/76	Datsun	0.16	0.78	0.82	245.21	10.9	89.7	86.7	29.15
		_								
171	6/3/76	Mercedes	0.08	0.61	0.78	259.77	10.3	86.7	86.3	29.15
172	6/4/76	Datsun	0.18	0.83	0.84	248.57	10.7		85.3	29.18
173	6/4/76	Mercedes	0.12	0.60	0.80	257.21	10.4	94.2	87.7	29.12
174	6/4/76	I.HPerkins	0.78	2.68	0.94	263.48	10.0	88.7	87.4	29.13
175	6/4/76	Peugeot	0.41	0.96	0.71	241.12	11.0	77.3	85.1	29.13
								İ		
176	6/4/76	Datsun	0.13	0.83	0.84	258.02		147.6	87.3	29.10
177	6/14/76	I.HPerkins	0.90	2.93	0.86	278.67	9.4	140.5	87.5	29.07
178	6/14/76	Peugeot	0.40	1.03	0.63	246.72	10.8	130.7	85.8	29.07
179	6/14/76	Mercedes	0.10	0.66	0.75	271.50	9.9	136.1	88.3	29.05
180	6/15/76	I.HPerkins	0.91	2.77	0.81	276.41	9.5	158.1	87.0	29.06
								i		
181	6/15/76	Mercedes	0.17	0.67	0.76	281.50	9.5	150.9	88.2	29.06
182	6/15/76	Datsun	0.15	0.85	0.85	268.25	10.0	141.4	86.3	29.06
183	6/15/76	Peugeot	0.43	0.56	0.56	245.40	10.8	147.5	87.1	29.07

# APPENDIX B

COEFFICIENTS OF EQUATIONS RELATING DEPENDENT AND INDEPENDENT VARIABLES

TABLE B-1. COEFFICIENTS OF HYDROCARBON (HC) EQUATIONS

"Independent"			Coef	ficient by	"independ	lent" varia	ble	
variable(s)	Vehicle	Constant	Н	Т	H <sup>2</sup>	T <sup>2</sup>	HT	r <sup>2</sup>
н	Datsun	0.155	-1.53E-4					0.020
<b></b>	I.HPerkins	0.503	2.16E-3					0.576
	Mercedes	0.134	-1.46E-5					0.000
	Peugeot	0.421	-5.26E-4					0.078
T	Datsun	0.255		-1.39E-3				0.046
_	I.HPerkins	0.037		7.86E-3		Í		0.235
	Mercedes	0.193		-7.51E-4			1	0.012
	Peugeot	0.679		-3.71E-3				0.012
н,н <sup>2</sup>								
н,н-	Datsun	0.126	9.14E-4		-6.95E-6			0.085
	I.HPerkins	0.529	1.21E-3		6.22E-6			0.584
	Mercedes	0.120	5.10E-4		-3.43E-6			0.012
	Peugeot	0.381	1.02E-3		-1.03E-5			0.127
$_{\mathrm{T},\mathrm{T}^{2}}$	Datsun	-1.821		5.18E-2		-3.38E-4		0.031
	I.HPerkins	4.537		-1.08E-1		7.39E-4		0.295
	Mercedes	-2.505		6.82E-2		-4.37E-4		0.115
	Peugeot	-2.219		7.09E-2		-4.76E-4		0.165
H,T	Datsun	0.246	-4.97E-5	-1.24E-3				0.047
·	I.HPerkins	0.351	1.97E-3	2.09E-3				0.588
	Mercedes	0.200	5.36E-5	-8.85E-4				0.013
	Peugeot	0.634	-2.73E-4	-2.89E-3				0.128
н,н <sup>2</sup> ,т,т <sup>2</sup>	Datsun	-1.577	6.37E-4	4.48E-2	-4.40E-6	-2.91E-4		0.153
11/11 /1/1-	I.HPerkins	3.570	1.71E-3	-8.06E-2	1.29E-6	5.28E-4		0.621
	Mercedes	-2.569	a	7.00E-2	5.51E-7	-4.50E-4		0.119
	Peugeot	-1.711	6.93E-4	5.66E-2	-6.39E-6	-3.79E-4		0.194
н,т, нт	Datsun	0.124	1.85E-3	2.76E-4			-2.31E-5	0.063
n,i, ni	I.HPerkins	0.124	4.57E-5	6.66E-4			2.31E-5	0.591
	Mercedes	0.466	2.20E-3	6.53E-4			-2.57E-5	0.391
	Peugeot	-0.022	1.08E-2	5.27E-3			-2.5/E-5 -1.34E-4	
	reugeot	-0.022	1.00E-2	3.2/E-3			-I.34E-4	0.330

a F-level (significance) insufficient for inclusion

TABLE B-2. COEFFICIENTS OF CARBON MONOXIDE (CO) EQUATIONS

Independent"				ficient by	"independ	ent" varia		
variable(s)	Vehicle	Constant	Н	Т	н2	<u>T</u> 2	HT	r2
••	Dat	0.770	2 047 4		•			,
Н	Datsun	0.779	2.84E-4					0.0
	I.HPerkins	1.708	7.08E-3					0.6
	Mercedes	0.639	-4.23E-5					0.0
	Peugeot	0.996	2.20E-5					0.0
T	Datsun	0.838		-4.47E-4				0.0
	I.HPerkins	-0.281		3.15E-2				0.4
	Mercedes	0.795		-1.97E-3				0.1
	Peugeot	1.385		-4.84E-3				0.2
H,H <sup>2</sup>	Datsun	0.777	3.71E-4		-5.66E-7			0.0
11,11	I.HPerkins	1.843	2.07E-3		3.29E-5			0.7
	Mercedes	0.663	-9.15E-4		5.70E-6			0.0
	Peugeot	0.990	2.64E-4		-1.61E-6			0.0
	reugeot	0.990	2.046-4		-1.016-0			10.0
$T,T^2$	Datsun	2.029		-3.10E-2		1.94E-4		0.0
	I.HPerkins	11.693		-2.77E-1		1.97E-3		0.4
	Mercedes	1.132		-1.06E-2		5.46E-5		0.1
	Peugeot	1.775		-1.49E-2		6.41E-5		0.2
н,т	Datsun	0.910	4.32E-4	-1.77E-3				0.0
, -	I.HPerkins	0.638	5.77E-3	1.46E-2				0.7
	Mercedes	0.812	1.36E-4	-2.31E-3				0.1
	Peugeot	1.485	6.03E-4	-6.65E-3				0.3
н,н <sup>2</sup> ,т,т <sup>2</sup>	Datsun	1.815	4.12E-4	-2.50E-2	a	1.48E-4		0.1
11,11 ,1,1	I.HPerkins	7.954	3.34E-3	-1.71E-1	1.55E-5	1.18E-3		0.7
	Mercedes	0.876	-1.05E-3	-2.71E-3	7.97E-6	a		0.2
	Peugeot	1.501	2.77E-4	-6.75E-3	2.22E-6	a		0.3
	Leageor	1.501	2.,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,	] 3.732 3	1	1		
H,T, HT	Datsun	0.828	1.70E-3	-7.60E-4			-1.53E-5	0.0
	I.HPerkins	1.506	-8.74E-3	3.86E-3			1.76E-4	0.7
	Mercedes	1.000	-3.05E-3	-4.59E-3			3.82E~5	0.2
	Peugeot	1.391	2.18E-3	-5.48E-3			-1.92E-5	0.3

a F-level (significance) insufficient for inclusion

TABLE B-3. COEFFICIENTS OF NITROGEN OXIDES (NO $_{\mathbf{X}}$ ) EQUATIONS

'Independent"		<u></u>	Coef	ficient by	"independ	lent" varia	able	
variable(s)	Vehicle	Constant	Н	Т	H2	т2	HT	r <sup>2</sup>
н	Datsun	1.039	-1.95E-3					0.57
	I.HPerkins	1.057	-1.73E-3					0.48
	Mercedes	0.983	-1.99E-3					0.60
	Peugeot	0.848	-1.65E-3					0.61
T	Datsun	1.180		-3.68E-3				0.05
	I.HPerkins	1.145		-2.72E-3				0.03
	Mercedes	1.160		-4.01E-3				0.07
	Peugeot	0.982		-3.24E-3				0.06
н,н <sup>2</sup>	Datsun	1.161	-6.43E-3		2.92E-5			0.77
	I.HPerkins	1.126	-4.28E-3		1.68E-5			0.56
	Mercedes	1.044	-4.23E-3		1.46E-5			0.65
	Peugeot	0.859	-2.08E-3		2.86E-6			0.61
т,т <sup>2</sup>	   Datsun	3.247		-5.67E-2		3.37E-4		0.07
-,.	I.HPerkins	3.078		-5.25E-2		3.17E-4		0.05
	Mercedes	3.797		-7.14E-2		4.27E-4		0.09
	Peugeot	-0.457		3.38E-2		-2.37E-4		0.07
н,т	Datsun	0.812	-2.21E-3	3.08E-3				0.60
	I.HPerkins	0.824	-2.01E-3	3.19E-3				0.52
	Mercedes	0.895	-2.08E-3	1.18E-3				0.61
	Peugeot	0.676	-1.85E-3	2.34E-3				0.63
н,н <sup>2</sup> ,т,т <sup>2</sup>	Datsun	2.998	-6.12E-3	-4.84E-2	2.62E-5	3.16E-4		0.78
	I.HPerkins	3.065	-3.93E-3	-5.26E-2	1.27E-5	3.52E-4		0.60
	Mercedes	1.023	-4.20E-3	a	1.42E-5	3.41E-6		0.65
	Peugeot	-0.233	-2.22E-3	2.61E-2	2.57E-6	-1.53E-4		0.64
H,T, HT	Datsun	1.612	-1.46E-2	-6.82E-3			1.50E-4	0.71
• •	I.HPerkins	1.364	-1.10E-2	-3.52E-3			1.09E-4	0.60
	Mercedes	1.512	-1.26E-2	-6.35E-3			1.26E-4	0.71
	Peugeot	0.885	-3.38E-3	-2.64E-4			4.28E-5	0.65

a F-level (significance) insufficient for inclusion

# APPENDIX C

STEPWISE MULTIPLE REGRESSION COMPUTER OUTPUTS

PAHIIAL

TULERANCE

SIGNIFICANCE

DEPENDENT VARIABLE.. HC

VARIABLE(S) ENTERED ON STEP NUMBER 3.. HSQ

MULTIPLE R .39171 ANALYSIS OF VARIANCE DF HEAN SQUARE F SIGNIFICANCE SUM OF SQUARES 1.767112 .155 R SQUARE REGRESSION .15343 ٧. .01334 .00333 STO DEVIATION . 114344 RESIDUAL 39. .0735R .00189

VARIABLE

STD ERPOR 8 BETA VARIABLE F SIGNIFICANCE ELASTICITY 6.8584751 TEMP .27821539E-01 .44778976E-01 2.5905144 .116 25.14272 GEH .63/12545E-03 .6568543NE-03 .94083143 .5904067 .33# .35113 -7.0217274 159 -.29116120E-#3 .17761386E-03 2.4872825 .109 -13.21661 HSQ -.43973713E-05 ,43097756E-05 1.0410628 -.6465061 . 314 -.24311 (CUNSTANT) -1.577378h 1.0803570 2.1317551

. 152

ALL VARIABLES ARE IN THE EQUATION.

48

VARIANCE/COVARIANCE MATRIX OF THE UNNORMALIZED REGRESSION COEFFICIENTS.

TEMP .00077 GEH -.00000 .00000 190 -.00000 .00000 ,00000 .00000 HS@ -.00000 .00000 -.00000 TEMP нза GEH 190

FILE HARE (CREATION DATE = 76/07/27.)
SUBFILE DATSUN

\*

DEPENDENT VARIABLE.. HC

## · BUHHARY TABLE

STEP	VARIABLE Entered removed	F TO Enter or remove	SIGNIFICANCE	MULTIPLE R	R SQUARE	R SQUARE CHANGE	SIMPLE R	OVERAĻL F	SIGNIFICANCE
1	TEHP	1,15711	.288	.21330	.04550	.04550	21330	1.01276	.37ë
	GEH	.06805	.796	.21697	.04708	.00158	14207		
5	TSU	3.85429	.057	.36170	.13083	.08375	22363	2.00694	.128
3	HSQ	1.04106	,314	.39170	.15343	.055P0	- '50101	1.76702	.155

----- VARIABLES NOT IN THE EQUATION -----

DEPENDENT VARIABLE..

VAPIABLE(S) ENTERED ON STEP NUMBER 2.. TSO

F SIGNIFICANCE MULTIPLE R .31916 ANALYSIS OF VARIANCE MEAN SQUARE DF SUM OF SUVARES .00405 R SOHARE REGRESSION 1.51214 .10186 Э. .01217 STO DEVIATION .00268 .05179 RESIDUAL 40. .10729

------ VARIABLES IN THE EQUATION ------VARIABLE BETA VARIABLE PARTIAL. TOLERANCE F STU EHROR B F SIGNIFICANCE SIGNIFICANCE ELASTICITY .00904 TEMP .60379665 HSG .05407 .3185B021E-02 -.24996369E-U1 .32168557E-01 -3.2657176 -2.50229 .955 .442 .3254779

3.0445363

1.19776

.04052

GEH .41239522E-03 .221272616-03 3,4735370 .070 150 .14800034E-03 .204777936-03 .52234702 . 474 (CUNSTANT) 1.8153172 2.0853721 1.2570737 .157

F-LEVEL OR IDLERANCE-LEVEL INSUFFICIENT FOR FURTHER COMPUTATION.

VARIANCE/CUVARIANCE MATRIX OF THE UNNORMALIZED REGRESSION COEFFICIENTS.

TEMP .00103

GEH .00000 .00000

-.00000 190 -.00001 .00000

> TEMP GEH TSU

50

FILE HARE (CREATION DATE = 76/07/27.)
SUBFILE DATSUN

DEPENDENT VARIABLE.. CO

## SUMMARY TABLE

STEP	VARIABLE	F TO	SIGNIFICANCE	MULTIPLE R	R SQUARE	R SQUARE	SIMPLE R	OVERALL F	SIGNIFICANCE
	ENTERED REMOVED	ENTER OR REMOVE				CHANGE			
1	TEMP	1.78832	.189	.05846	.00342	.00342	กรธรร	2.03077	.144
	GEH	3,90755	.055	.30022	.09013	.08672	.22460		
5	T 9 0	.522,35	.474	.31416	.10186	.01173	-,05330	1,51219	.256

SUSTILE DATSON

DEPENDENT VARIABLE. NOX

VARIABLE(S) ENTERED ON STEP NUMBER 3.. 180

MULTIPLE R	.88710	ANALYSIS OF VARIANCE	OF	SUM OF SHUARES	MEAN SQUARE	F S	SIGNIFICANCE
R SOUARE	. 78644	REGRESSION	٧.	- 34119	.04780	36.01274	.000
NOITAIVED DIE	.05211	RESIDUAL	39,	.10591	. nu 2 7 2		

	VARIA	RLES IN THE EQUA	rion			VARIABLES NOT	IN THE EQUA	T10N
VARIABLE	В	STO ERROR B	F	BETA	VARTABLE	PARTIAL	TULERANCE	F
			SIGNIFICANCE	ELASTICITY				SIGNIFICANCE
TEMP	48448727E-(il	.33778523E-01	2.1068349	-3.11128697				
			.155	-4.34415				
GEH	61215130E-02	.788052246-03	PDED4E, DB	-2.3719898				
	· <del>-</del> · · -		t	54445				

1.6126474

3.1875915 2.31775

.23426

ALL VARIABLES ARE IN THE EQUATION.

(CONSTANT) 2.9975814

.24232044E-04

.31610019E-03

VARIANCE/COVARIANCE MATRIX OF THE UNNORMALIZED REGRESSION COEFFICIENTS.

.51705962E-05

.21308387E-U3

1.2961440

25.738487

2.2005128

5.3485425 dsn.

.000

.146

TEMP .00111 GEH -.00001 .00000 130 -.00001 .00000 .00000 HSQ ,00000 -.00000 -.00000 , որույր TEHP GEH TSQ HBO

G

HSQ

TSQ

•

FILE HARE (CREATION DATE = 75/07/27.)
SUBFILE DATSUN

\*\*\*\*\*\*\*\*\*\*\*\*\*\*\*\*\*\* MULTIPLE REGRESSION \*\*\*\*\*\*\*\*\*\*\*

DEPENDENT VARIABLE.. NOX

## SUMPARY TABLE

STEP	VARIABLE	F TO	SIGNIFICANCE	MULTIPLE R	R SHUARE	R SQUARE	SIMPLE R	OVERALL F	SIGNIFICANCE
	ENTERED REMOVED	ENTER OR REMOVE				CHANGE			
1	TENP	2,97050	SP0.	.23553	.05547	.05547	-,23553	30.41644	.000
	GEH	56.12861	0	.77543	. PO 130	.54582	-,75658		
2	HSQ	30,85676	.000	,88030	.77492	17363	- 65138	45,90579	.000
3	TSQ	5.20021	.146	,88710	,78694	.01505	- 53105	36,01274	.000

----- VARIABLES NOT IN THE EQUATION ------

DEPENDENT VARIABLE..

VARIABLE(S) ENTERED ON STEP NUMBER 3.. HOU

SIGNIFICANCE HULTIPLE R .78780 ANALYSIS OF VARIANCE DF SUM OF SQUARES MEAN SQUARE P SQUARE . 65063 REGRESSION 4. .38547 .09637 16.35964 .000 STD DEVIATION .07675 RESIDUAL 40. .23562 .00589

----- VARIABLES IN THE EQUATION -----TOLERANCE F BETA VARIABLE PARTIAL VARIABLE ₿ STU ERROR B SIGNIFICANCE ELASTICITY SIGNIFICANCE

2.7459411 ~4.9721568 TEMP -.R0645504E-01 .48665007E-01 .105 -9.68929 GEH .17124652E-02 .10717326E-02 2.5531164 .6058778 .118 .19227 rso .52837929E-03 .31158Nh7E-03 2.8757462 5.1114408 5.10836 . D9R .12911756E-05 .699804226-05 .34042191E-01 .0719545 HSQ .01414 .855 (CUNSTANT) 3.5698818 1.8804978 3.6038109 .065

ALL VARIABLES ARE IN THE EQUATION.

VARIANCE/COVARIANCE MATRIX OF THE UNNORMALIZED REGRESSION COEFFICIENTS.

TEMP .00237 GEH -.00001 .00000

TSU -.00002 .00000 .00000

.00000 -.00000 .00000 HSQ -.00000

> TEMP GEH TSU HSQ

FILE HARE (CREATION DATE = 75/07/27.)
SUBFILE PERKINS

DEPENDENT VARIABLE.. HC

# SUMMARY TABLE

	STEP	VARIAB		F TO	SIGNIFICANCE	MULTIPLE R	H SQUARE	R SQUARE	SIMPLE R	OVERALL F	SIGNIFICANCE
		ENTERED R	EMOVED E	NTER OR REMOVE				CHANGE			
	1	TEMP		1.24971	.270	.48456	.23480	08465.	.48456	29.97685	.000
		GEH		36.01496	.000	.76684	.58805	,35325	.75881		
Uī	5	TSU		3.48362	.069	.78760	.62031	.03226	.49264	22.32754	0
ŭ	3	HSQ		.03404	.855	.78780	.62063	.00032	.75595	16.35964	.000

DEPENDENT VARIABLE.. CO

VARIABLE(S) ENTERED ON STEP NUMBER 3.. HSU

MULTIPLE R SUM OF SQUARES MEAN SQUARE SIGNIFICANCE ANALYSIS OF VARIANCE DF .89065 R SQUARE 4.39639 1.09710 38.36747 REGRESSION 4. .000 .79325 STU DEVIATION .16925 RESIDUAL 40. 1.14586 .02865

.062

TEMP -.17145556 .10731815 2.5524475 -3.5389190 -6.11899 .118 GEH .33359139E-02 .23634304E-02, 1.9922509 .3931507 1 .166 .11125 2.9696998 3.8345758 TSQ .11840857E-02 .68711097E-03 3.40030 .093 HSQ .15504634E-04 .154323RIE-04 1.0093858 .2892484 .321 .05042 (CONSTANT) 7.9542854 4.1469539 3.6791198

ALL VARIABLES ARE IN THE EQUATION.

VARIANCE/COVARIANCE MATRIX OF THE UNNORMALIZED REGRESSION COEFFICIENTS.

.01152 TEMP .00001 GEH -.00007 TSQ -.00007 .00000 .00000 HSQ onngo. .00000 -.00000 -.D0000 7 TEMP TSQ наа GEH

뗬

. FILE HARE (CREATION DATE = 76/07/27.)
SUBFILE PERKINS

DEPENDENT VARIABLE.. CO

## SUMMARY TABLE

STEP	VARIA Entered	BLE REMOVED	F TO Enter or remove	SIGNIFICANCE	MULTIPLE R	R SQUARE	R SQUARE CHANGE	SIMPLE R	OVERALL F	SIGNIFICANCE
1	TEMP		11.97484	.001	.65109	.42392	.42392	.65109	P8*10412	.000
3 5	GEH TSQ HSQ		60.66867 4.58385 1.00939	.000 .038 .321	.87426 .88771 .89065	.76433 .78803 .79325	.34042 .02370 .00522	.83495 .65815 .84841	50.80854 38,36747	0 • 000

----- VARIABLES NOT IN THE EQUATION -----

DEPENDENT VARIABLE.. NOX

VARIABLE(S) ENTERED ON STEP NUMBER 3.. TSO

MULTIPLE R MEAN SQUARE .77710 ANALYSIS OF VARIANCE DF SUM OF SQUARES SIGNIFICANCE R SQUARE REGRESSION .07134 .60389 4. .28535 15,24547 .nou STD DEVIATION .06841 RESIDUAL 40. .18717 .00468

VARIABLE B STD ERROR B F HETA VARIABLE PARTIAL TOLERANCE F

SIGNIFICANCE ELASTICITY SIGNIFICANCE

TEMP 1.4714049 -.52612743E-01 .43373539E-01 -3,7191462 .232 -4.52466 GEH -.39292019E-02 16.920753 -1.5859219 .95520042E-03 -.31576 .000 HSQ .12683808E-04 .62371274E-05 4.1355196 .8103868 PEPPO. .049 .35227266E-U3 1.6091656 3.9070267 TSG .27770172E-U3 2.43770 .212 3.3453072 (CONSTANT) 3.0654829 1.6760246 .075

----- VARIABLES IN THE EQUATION ------

ALL VARIABLES ARE IN THE EQUATION.

VARIANCE/COVARIANCE MATRIX OF THE UNNORMALIZED REGRESSION COEFFICIENTS.

TEMP .00188
GEH -.00001 .00000
TSQ -.00001 .00000 .00000
HSU .00000 -.00000 .00000

TEMP GEH TSQ HSQ

FILE HARE (CREATION DATE = 76/07/27.)
SUBFILE PERKINS

DEPENDENT VARIABLE.. NOX

## SUMMARY TABLE

STEP	VAR	IABLE	F 10	SIGNIFICANCE	HULTIPLE R	R SQUARE	R SQUARE	SIMPLE R	OVERALL F	SIGNIFICANCE
	ENTERED	REMOVED	ENTER OR REMOVE				CHANGE		·	
1	TEMP		3.29449	.077	.19195	.03684	.03684	19195	<b>₫3.0587</b> ¥	.000
	GEH		42.87088	.000	.72344	.62336	. 48652	69712		
2	HSU		6.42699	.015	.76678	.58795	.06459	59496	19.50117	0
3	TSO		1,60917	.212	.77710	.60389	.01594	18772	15.24547	.000

DEPENDENT VARIABLE.. HC

VARIABLE(S) ENTERED ON STEP NUMBER 3.. HSQ

MULTIPLE R .34525 ANALYSIS OF VARIANCE I) F SUM OF SQUARES MEAN SQUARE F SIGNIFICANCE R SQUARE .11920 REGRESSION ٧. .01274 .0031H 1.28563 . 243 STD DEVIATION .00248 .04977 RESIDUAL 38. .09413

----- VARIABLES IN THE EQUATION ------- VARIABLES NOT IN THE EQUATION ------

VARIABLE B STD ERROR B F BETA VARIABLE PARTIAL TOLERANCE F

SIGNIFICANCE ELASTICITY SIGNIFICANCE

TEMP .70926714E-01 . 34834746E-01 4,1459048 10.5484604 .049 43.08841 GEH -.652658986-04 .R0198282E-03 . 46536450E-05 -.0536894 . 976 -.03637 rsu -.455651476-03 .22132731E-03 4.2383373 -10.3475947 -22.50923 .046 HSQ .33930900E=01 .96139380E~n6 .52191962E-05 .1249337 .05183 .855 (CONSTANT) -2.6020518 1.3538498 3.6939491 .065

ALL VARIABLES ARE IN THE EQUATION.

VARIANCE/COVARIANCE MATRIX OF THE UNNORMALIZED REGRESSION COEFFICIENTS.

TEMP .00121 GEH -.00001 .00000 TSQ -.00001 .00000 .00000

00000 - 00000.- 000000, O00000

TEMP GEH TSW HSW

FILE HARE (CREATION DATE = 76/07/27.)
SUBFILE MERCEDES

DEPENDENT VARIABLE.. HC

# SUMMARY TABLE

STEP	VARIABLE	F TO	SIGNIFICANCE	MULTIPLE R	R SQUARE	R SQUARE	SIMPLE R	OVERALL F	SIGNIFICANCE
	ENTEKED REMOVED	ENTER OR REMOVE				CHANGE			
1	TEMP	.53481	.469	.10847	.01177	.01177	10847	.27034	.765
	GEH	.06368	\$08	.11549	<b>PEE10.</b>	.00157	-,01204		
2	T 5 G	4.64834	,037	.34411	.11841	,10507	-,11878	1,74611	.173
3	HSQ	EPEEO.	.855	.34525	,11920	.00079	- 03836	1.28563	.293

FILE HARE (CREATION DATE = 76/07/27.)
SUBFILE MERCEDES

DEPENDENT VARIABLE.. CO

VARIABLE(S) ENTERED ON STEP NUMBER 2.. HSQ

ANALYSIS OF VARIANCE SIGNIFICANCE D۴ SUM OF SQUARES MEAN SQUARE MULTIPLE R .49193 .00544 R SQUAPE . REGRESSION Э. .01631 4.15033 •015 .24200 34. .05109 .00131 STD DEVIATION .03619 RESIDUAL

----- VARIABLES NOT IN THE EQUATION ---------- VARIABLES IN THE ENUATION -----F VARIABLE F BETA VARIABLE PARTIAL TOLERANCE В STO ERROR H SIGNIFICANCE SIGNIFICANCE ELASTICITY .24845263 TEMP 9.6680483 ₹EnneP#. .. 180 -.27097443E-02 En-34P58#158. -.BBUPO 100001 .621 .003 -. 34 356 GEH **1.6507846** -1.0913775 -.105364536-02 .55144391E-03 -.12255 . D63 1,3038171 HSG 5,0237772 .74682937E-05 .35550863E-05 .08465 .031

F-LEVEL OR TOLERANCE-LEVEL INSUFFICIENT FOR FURTHER COMPUTATION.

VARIANCE/CUVARIANCE MATRIX OF THE UNNORMALIZED REGRESSION COEFFICIENTS.

.70822536E-01

152.42545

U

TEMP .00000

(CONSTANT)

62

GEH .00000 .00000

.87581296

HSQ -.00000 -.00000 .00000

TEMP GEH HSG

FILE HARE (CREATION DATE = 76/07/27.)
SUBFILE MERCEDES

DEPENDENT VARIABLE.. CO

# SUMMARY TABLE

STEP	VARIABLE	F TO	SIGNIFICANCE	MULTIPLE R	R SQUARE	R SQUARE	SIMPLE R	OVERALL F	SIGNIFICANCE
	ENTERED REMOVED	ENTER OR REMOVE				CHANGE		•	
1	TEMP	6.65869	.014	.35829	.12837	.12837	35829	3.37418	.044
	GEH	.74718	.393	.37994	.14435	.01598	04379		
2	HSO	5.02378	.031	.49193	.24200	.09764	.01347	4.15033	.012

FILE HARE (CREATION DATE = 76/07/27.)
SUBFILE MERCEDES

annanananananananananan MULTIPLE RE'GNESSION annananananananananan

DEPENDENT VARIABLE.. NOX

VARIABLE(8) ENTERED ON STEP NUMBER 2.. HSQ

MULTIPLE R	.81193	ANALYSIS UF VARIANCE	DF	SUM UF SQUARES	MEAN SQUARE	F SIGNIFI	CANCE
R SQUARE	.65920	REGRESSION	3.	<b>₽</b> ח₽ט <b>E</b> •	.10301	25.14601	Ü
STD DEVIATION	-06400	RESIDUAL	99.	. 15977	.00910		

F SIGNIFICANCE	BETA	VARIABLE	PARTIAL	TOLERANCE	F SIGNIFICANCE
.764 .764 .764 .000 .000 .143008 .244 .446	.04502 .04502 .04502 .0450645 .045488 .0864	150	.21180	.00041	1.7847601 .190
	.000 5.1430081	37224 5.1430081 .8845487 .024 .12204 64.763867	3724 5.143081 .8845487 .024 .1284 64.763867	3724 5.143081 .8845487 .024 .1284 64.763867	.00037224 5.1430081 .8845487 .024 .12604 64.763867

F-LEVEL OR TOLERANCE-LEVEL INSUFFICIENT FOR FURTHER COMPUTATION.

VARIANCE/COVARIANCE MATRIX OF THE UNNORMALIZED REGRESSION COEFFICIENTS.

TEMP .00000

64

GEH .00000 .00000

00000 -.00000 .00000

TEMP GEH HSQ

FILE .HARE (CREATION DATE = 75/07/27.)
SUBFILE MERCEDES

DEPENDENT VARIABLE.. NOX

## SUMMARY TABLE

STEP	VARI ENTERED	ABLE REMOVED	F TO Enter or remove	SIGNIFICANCE	MULTIPLE H	R SQUARE	R SQUARE CHANGE	SIMPLE R	OVERALL F	SIGNIFICANCE
1	TEMP GEH	,	.55806 55.75084	.459 .000	.27683 .78375	.07663 .61426	.07663 .53763	27683 78031	31,84876	.000
5	HSQ		5.14301	P50.	19118.	.65920	.04494	70206	25,14601	0

DEPENDENT VARIABLE. HC.

VARIABLE(S) ENTERED ON STEP NUMBER 3.. HSQ

SIGNIFICANCE MULTIPLE R .44068 ANALYSIS OF VARIANCE DF SUM OF SUUARES MEAN SQUARE 45855.5 .085 R SQUARE .14450 REGRESSION ٧. .05126 .01282 STD DEVIATION .07582 RESIDUAL 37. .21270 .00575

VARIABLE B STO ERROR B F BETA VARIABLE PARTIAL TOLERANCE F
SIGNIFICANCE ELASTICITY
SIGNIFICANCE

. 384

SIGNIFICANCE ELASTICITY TEMP .5664N379E-N1 .50273304E-01 1.26933RS 5.1217971 11.91522 .267 .3676753 .69271042E-03 .11n45447E-02 .34331168 GEH .13867 .534 -5.3683950 1.3860888 -.37868483E-03 .351P#450E-03 TSQ .247 -6.43520 .78108884 -.5296585 -.63935512E-05 .72342237E-05 HSQ .383 -.12732 .77497266 (CONSTANT) -1.7109986 1.4435981

ALL VARIABLES ARE IN THE EQUATION.

VARIANCE/COVARIANCE MATRIX OF THE UNNURMALIZED REGRESSION COEFFICIENTS.

TEMP .00253

GEH -.00001 .00000 TSQ -.00002 .00000 .0000

00000. 00000.- 000ud.- 000ud.

TEMP GEH TSQ HSQ

76/07/27. PAGE &

FILE HARE (CREATION DATE = 76/07/27.)
SUBFILE PEUGEOT

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DEPENDENT VARIABLE.. HC

## SUMMARY TABLE

STEP	VARIA	BLE	, <b>f to</b>	SIGNIFICANCE	HULTIPLE R	R SQUARE	R SQUARE	SIMPLE R	OVERALL F	SIGNIFICANCE
	ENTERED	REMOVED	ENTER OR REMOVE				CHANGE			
1	TEMP		2,25686	.141	,33589	.11585	,11282	-,33589	2,86970	,069
	GEH		,69172	.411	.35817	.12824	.01546	27400		
2	TSU		2,25842	.141	.42094	.17719	.04890	-,34346	2,72767	, 057
3	HSQ		,78109	.38€.	.44068	.19420	.01701	32723	5.55454	.005

DEPENDENT VARIABLE.. CO

VARIABLE(S) ENTERED ON STEP NUMBER 2.. HSQ

MULTIPLE R ANALYSIS OF VARIANCE SUM OF SQUARES MEAN SQUARE . 61588 DF SIGNIFICANCE R SQUARE REGRESSION .37563 .07105 . 112368 7.62031 Э. .000 STO DEVIATION .05575 RESIDUAL 30. .11811 .00311

----- VARIABLES IN THE EQUATION ---------- VARIABLES NOT IN THE EQUATION -----VARIABLE 8 F VARIABLE PARTIAL TOLERANCE STD ERROR B BETA ------SIGNIFICANCE SIGNIFICANCE ELASTICITY TEMP -.67511511E-U2 .14762319E-03 .14155183E-02 850747.55 -.7211537 TSQ -.00200 .00105 .000 -.54233 .490 GEH .27687967E-03 ,78451551E-03 .12456014 .1736031 .726 .02117 HSQ .22228480E-05 .50901763E-05 .19070140 .2175291

.01690

.665

0

184.86129

F-LEVEL OR TOLERANCE-LEVEL INSUFFICIENT FOR FURTHER COMPUTATION.

.11037313

VARIANCE/CUVARIANCE MATRIX OF THE UNNORMALIZED REGRESSION COEFFICIENTS

TEMP .00000

(CONSTANT)

GEH -.00000 .00000

1.5006740

00000 -.00000 -.00000 .00000

TEMP GEH HSR

FILE HARE (CREATION DATE = 76/07/27.)
SUBFILE PEUGEOT

DEPENDENT VARIABLE.. CO

## SUMMARY TABLE

STEP	VARIABLE Entered removed	F TO Enter or remove	SIGNIFICANCE	MULTIPLE R	R SQUARE	R SQUARE Change	SIMPLE R	OVERALL F	SIGNIFICANCE
1	ТЕНР	23.13880	.000	.51675	.26703	.26703	51675	11.57532	.000
2	GEH HSQ	6.55446 .1907N	.014	.61588 .61035	.37249 .37563	.10546	.005P#	7,62031	•000

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SIGNIFICANCE

FILE HARE (CREATION DATE = 76/07/27.)
SUBFILE PEUGEOT

DEPENDENT VARIABLE... NOX

VARIABLE(S) ENTERED ON STEP NUMBER 3.. HSQ

MULTIPLE R .80181 ANALYSIS OF VARIANCE DF SUM OF SQUARES MEAN SQUARE SIGNIFICANCE R SQUARE .21290 .64289 REGRESSION ٧. .05323 16.65255 .000 STD DEVIATION .05654 RESIDUAL 37. .11826 .00320

VARIABLE B STD ERROR B F BETA VARIABLE PARTIAL TULERANCE F

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SIGNIFICANCE ELASTICITY TEMP .26146703E-01 .37486275E-N1 .48650722 2.1108802 .490 5.90066 GEH -.22215275E-02 .8236N344E-03 7.2755710 -1.0527248 -.23453 .010 TSQ -.15287658E-03 .239837636-03 .40630040 -1.9348974 .528 -1.37003 HSQ .25647357E-05 .53941968E-05 .1900608 .22694642 .02699 .637 EPTPEEES.- (INATEMOD) 1.4492434 .25936498E-01 .873

ALL VARIABLES ARE IN THE EQUATION.

VARIANCE/COVARIANCE MATRIX OF THE UNNORMALIZED REGRESSION COEFFICIENTS.

.00141 TEHP GEH -.00001 .00000 150 -.00001 .00000 .00000 HSO .00000 -.00000 -.00000 .00000 TEMP GEH TSO 115Q

FILE HARE (CREATION DATE = 76/07/27.)
SUBFILE PEUGEOT

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DEPENDENT VARIABLE.. NOX

#### SUMMARY TABLE

STEP	VARIABLE Entered removed	F TO Enter or remove	SIGNIFICANCE	MULTIPLE R	R SQUARE	R SQUARE Change	SIMPLE R	OVERALL F	SIGNIFICANCE
1	TEMP	2.83142	.100	.26195	.06865	.08865	26195	34.37877	.000
	GEH	61.36343	Ü	.79880	.63808	.56946	78218		
2	T S Q	.27768	.601	.80044	.64070	.00263	26524	42.58726	.000
3	HSQ	.22695	.637	.00101	.64289	.00219	74079	16.65255	.000

ETLE MARE (COMAIN) DATE = 77703/24.)  MULTIPLE PEGRESSION  MEAN SQUARE  F SUM OSSAY												
DEFFINITY VARIABLE. HC  VARIABLES ARE IN THE EDUATION.  MULTIPLE PEGHESSION  ANALYSIS OF VARIANCE OF SUM OF SOURCES HEAN SOURCE POLICY AND ANALYSIS OF VARIANCE OF SUM OF SOURCES AND ANALYSIS OF VARIANCE OF SUM OF SOURCES AND ANALYSIS OF VARIANCE OF SUM OF SOURCE POLICY AND ANALYSIS OF VARIANCE OF SUM OF SOURCE POLICY AND ANALYSIS OF VARIANCE OF SUM OF SOURCE POLICY AND ANALYSIS OF VARIANCE OF SUM OF SOURCE POLICY AND ANALYSIS OF VARIANCE OF SUM OF SOURCE POLICY AND ANALYSIS OF VARIANCE POLICY AND ANALYSIS		EATION DATE = 7	7/03/24.)									
DEPFMENT VARIABLE. HC  VANIARIF(S) FRIERED ON SIFP NUMBER 2. HT  HILITPLE R .PSDZE ANALYSIS OF VARIANCE OF SUL OF SQUARES HEAN SQUARE F SIGNIFICANCE ARSOLUTE	atmr (LF marana											
WANTARLE(R) FRIERED ON RIFP NUMBER 2. HT  HILTIPLE R .2502P ANALYSIS OF VARIANCE DF RUM OF SQUARES HEAN SQUARE F SIGNIFICANCE R SQUARE .05250 QCGRSRSION 3. 40559 .00181 .85101 .553 ADJURITOR R SQUARE .07556 QCGRSRSION 3. 40559 .00181 .85101 .553 RID DEVIATION .01513 COFF OF VARIABILITY 31.6 PCT	* * * * * * * * * *	* * * * * * * *	* * * * * M		PLE	8 E G H I	9 8 1 0	N * * * *	* * * *	* * * * * *	* * * * * *	* * * *
MILITIZER .2502P ANALYSIS OF VARIANCE DF SUM OF SQUARES HEAN SQUARE F SIGNIFICANCE N SQUARE N	DEPENDENT VARTABLE.	• нс										
N SQUARE	VANTABLE (9) FRIERED	ON STEP NUMBER	P HT									
N SQUARE												
VARIABLES IN THE EQUATION VARIABLE NOT IN THE EQUATION VARIABLE  VARIABLE  B  STD ERROR R  F  STGNIFICANCE ELASTICITY  SIGNIFICANCE  GEH ,18524440E-n2 ,23420098E-02 ,62562311 1;7166090 TEMP ,27560788E-03 ,21916166E-02 ,1581407E-03 1,02078  HT -,23054379E-04 ,28290393E-04 ,66407298 -1,0445679  -(CONSTANT) - ,12394465 ,17344106 ,51038411 .274  ALL VARIABLES ARE IN THE EQUATION.	R SQUARE ADJUSTED R SQUARE	.06264	REGRESSION RESIDUAL		3. 40.	-	.00544 .0814.7		.00181 ,00504			
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FILE HARE (CREATION DATE = 77/03/24.) SUBFILE MERCEDES DEPENDENT VARIABLE .. VARIABLE(S) ENTERED ON STEP NUMBER 2.. HT MULTIPLE R .84429 ANALYSIS OF VARIANCE SUM OF SQUARES MEAN SQUARE F SIGNIFICANCE A BOHANE .712R3 REGRESSION ₹. .3341R .11139 37.26427 ADJUSTED R SOUARE .69074 RESIDUAL 39. . 1 3463 .00345 STD DEVIATION .05875 COEFF OF VARIABILITY 7.0 PC [ ----- VARIABLES NOT IN THE EQUATION -----VARIABLE STD ERROR B HFTA - -VARIABLE PAPTIAL TOLERANCE  $\infty$ ίœ **BIGNIFICANCE** ELASTICITY SIGNIFICANCE GEH -: 12587118E-A1 .5885P8P8E-US 19\_065916 -4:9437421 -.000 - - -1.11391 TEMP -.43503231E-n2 .24814212E-NP 6.5492362 ··· EEPRRE#:- · · -.41259 13.385557 .12598231E-n3 40-39064EFFE:-91895 .001 · (CONSTANT) 1468861 57.763839 f) F - ALL VARTABLES ARE IN-THE EQUATION: ---

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77/03/24. PAGE 5

FILE HARE (CREATION DATE = 77/03/24.) SUBFILE PEUGEOT \* MULTIFIE REGRESSION \*\*\*\*\*\*\*\*\*\*\*\*

DEPENDENT VARIABLE.. HC

SUMMARY TABLES ........

} } .	STEP	VARIABLE Entered removed	F TO ENTER OR REMOVE	SIGNIFICANCE	MULTIPLE R	H SQUARE	R SQUARE CHANGE	SIMPLE R	OVERALL F	SIGNIFICANCE
Γ	ì	GEH	.69172	.411	.27900	.07784	.07784	27900	2.86970	.069
ι	_	TEMP	2.25686	.141	.35817	12829	05044	33589	4	
ĺ	6	· HT	11,40367	.noa	.57402	.32950	\$0191	33149	6.22470	• 005

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FILE HARE (CREATION DATE = 77/03/24.)
SUBFILE PEUGEOT

DEPENDENT VARIABLE.. CO

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STEP	VARIABLE ENTERED REMOVED	F TO Enter or remove	SIGNIFICANCE	MULTIPLE R	R SQUARE	R SUIJARE CHANGE	SIMPLE H	OVERALL F	SIGNIFICANCE
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FILE HARE (CREATION DATE = 77/03/24.) SUBFILE PEUGEOT \* \* \* \* \* \* \* \* HULTIPLE - REGRESSION \* \* \* \* \* \* \* \* DEPENDENT VARIABLE.. Nox -SU-MMARY TABLE - STEP VARIABLE F TO SIGNIFICANCE MULTIPLE H R SQUARE R SQUARE SIMPLE R OVERALL F SIGNIFICANCE ENTERED REMOVED ENTER OR REMOVE CHANGE 61.36343 (I .78218 .61180 -.7H21A 34.37877 H30 I .61180 .000 TEMP 2.83142 .100 .79880 .63808 .02628 -.26195 1.79144 .189 .80843 .65437 P5410. -.75075 23.48143 HT .000 Ω...

	TECHNICAL REPORT DATA (Please read Instructions on the reverse before c	ompleting)		
1. REPORT NO.	2.	3. RECIPIENT'S ACCESSIONNO.		
EPA 600/2-77-116				
4. TITLE AND SUBTITLE		5. REPORT DATE		
LIGHT-DUTY DIESEL EMISSIO	N CORRECTION FACTORS FOR	July 1977		
AMBIENT CONDITIONS	6. PERFORMING ORGANIZATION CODE			
7. AUTHOR(S)		8. PERFORMING ORGANIZATION REPORT NO.		
Charles T. Hare		6.1 ERFORMING ORGANIZATION REPORT NO.		
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9. PERFORMING ORGANIZATION NAME	AND ADDRESS	10. PROGRAM ELEMENT NO.		
Southwest Research Instit	lAA601 BC-09 (FY-77)			
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15. SUPPLEMENTARY NOTES				

#### 16 ABSTRACT

Since emission measurements from passenger cars are performed at one standard set of ambient conditions and since emission rates of HC, CO, and NO are sensitive to temperature and humidity, it is necessary to determine the influence of ambient conditions on emissions from major classes of vehicles. Although such information has been available for gasoline engine powered cars for sometime, no such data were available for diesel powered passenger cars.

This report indicates that diesel HC and CO emissions are relatively insensitive to ambient conditions. Diesel NO emissions, however, are sensitive to humidity but to a smaller extent than gasoline engines. Humidity correction factors for NO emissions also appear to vary with vehicle power-to-weight ratios and are greater for higher powered vehicles.

17.	KEY WORDS AND DOCUMENT ANALYSIS
n. DESCRIPTORS	b.IDENTIFIERS/OPEN ENDED TERMS c. COSATI Field/Group
*Air pollution	13B
*Automobiles *Diesel Engines	13F 21G
*Exhaust emissions	21B
*Correction	O4B
*Temperature *Humidity	
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